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THE SHIP STRUCTURE COMMITTEE is constituted to prosecute a research program to improve the hull structure of ships and other marine structures by an extension of knowledge pertaining to design, materials and methods of construction

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An Interagency Advisory Committee Dedicated to the Improvement of Marine Structures

August 2, 1990

SSC-334 SR-1305

### INFLUENCE OF WELD POROSITY ON THE INTEGRITY OF MARINE STRUCTURES

In the marine industry, we are concerned with the quality of weldments and the effect of weld defects on the strength and integrity of marine structures. This report is intended to provide a better understanding of the influence of weld metal porosity on the integrity of marine structures by examining the effects of porosity on fatigue resistance of ship steel weldments.

Rear Admiral, U. S. Coast Guard Chairman, Ship Structure Committee

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predictive model was developed crack initiation and propagate porosity, co-linear porosity. The models sensitivity to the stress, pore size and porosity. A variable amplitude load data, and used to predict act and the main conclusion of the fifthe weld reinforcement is type and size of porosity withistory the welds were not puriously, the results are relatives.	trol the fatigue life of ed incorporating those pation. Four types of portion. Four types of portion, and cluster porosity, ed parameters (plate thickly type) was studied and adding history was develoted service lives. The study was that poroside to intact. If the result control fatigue life, redicted to fail during atted to the American Bur	welds containing porosity. A arameters to account for both osity; single pore, uniform were examined using the model. kness, stress ratio, residual discussed. ped from SL-7 loading history ity is of little concern in welds inforcement is removed, the When subjected to the service
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# STUDY TO DETERMINE THE INFLUENCE OF WELD POROSITY ON THE INTEGRITY OF MARINE STRUCTURES

by

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### 1. INTRODUCTION

The objective of this study is to obtain a better understanding of the influence of weld porosity on the integrity of marine structures. Understanding the effects of porosity on the mechanical properties of weldments is important for the safe design of welded marine structures. Information on the porosity effects for a weldment would be useful in specifying welding processes and procedures. The expected service conditions of a weld could dictate the amount of porosity allowed. A welding process which would be expected to result in porosity levels corresponding to that allowable amount could be rationally determined and specified. The inspection and maintenance of welded structures would also benefit from a refined understanding of the detrimental effects of various sizes, shapes, and patterns of porosity.

Previous investigations on the effects of weld porosity on integrity of structures indicate that there is very little influence of porosity upon brittle fracture properties  $^{[1]}$ . However, porosity has been shown to influence the fatigue properties of welds  $^{[1-7]}$ . The motivation for the present study comes from the potential of modern fatigue technology and fracture mechanics principles to analytically predict the fatigue performance of weldments. The literature provides sufficient information on the dependence of fatigue performance on parameters such as size of pores, number of pores, pore shape and pattern. These parameters will be incorporated into a fatigue life estimation model based upon fatigue and fracture concepts.

### 2. DISCUSSION OF THE PROBLEM

### 2.1 Limits of Concern

The results of most of the studies examining the effects of porosity conclude that porosity does not effect the mechanical properties of a weldment unless the amount of porosity is extremely large  $\begin{bmatrix} 1-5 \end{bmatrix}$ . Regarding fatigue, the most critical location for a weld is generally the weld toe. This abrupt change in geometry from the weld metal reinforcement to the base metal results in a stress concentration and acts as a fatigue crack initiation site. Pores are, by comparison, much less severe stress concentrations.

The severity of the weld-toe stress concentration decreases with decreasing weld reinforcement size. That is, the smaller the weld reinforcement, the less effect the weld toe will have in initiating a fatigue crack. This fact suggests that if the weld reinforcement is shallow enough, the stress concentration due to the weld toe will be less than that resulting from a pore. The pore would then be the critical location for fatigue.

Consider the following example. The stress concentration factor,  $K_{t}$ , for a pore in an infinite body subjected to an axial stress is 2.05 (for Poisson's ratio of 0.3). The stress concentration factor for the toe of a butt weld subjected to axial tension [8] is 3.06 for a 0.5 inch thick plate, having a reinforcement width of 0.29 inch (60 degree bevel) and height of 0.17 inch, and a weld toe radius of 0.02 inch. This means that if a pore ( $K_{t}$  = 2.05) were present in the weld, the more highly stressed location would still be the weld toe ( $K_{t}$  = 3.06). The reinforcement height at which the stress concentrations would be equal for both the weld toe and the pore is 0.11 inch. At this reinforcement height, there would be an equal chance of a fatigue crack initiating at the toe or at the pore. At heights below this value, the fatigue crack would be expected to initiate at the pore.

This example is an over simplification of a rather complex stress analysis problem. Factors such as bending stress, almost always

present in actual service, and difficulty in accurately measuring the weld toe radius have not been considered. Both of these effects would increase the weld-toe stress concentration. The example does illustrate, however, that unless the weld reinforcement is shallow, fatigue cracks would not be expected to initiate from a pore.

### 2.2 Factors of Concern

Having discussed the fact that weld porosity is generally only a problem when the weld reinforcement is shallow or removed, or when porosity is excessive, the factors that must be addressed in analyzing this specific problem will be outlined.

### 2.2.1 Fracture Mechanics

Porosity can be characterized as a blunt defect having no sharp asperities which can be analyzed as cracks. Since cracks do initiate from pores, at some point in the cracks growth, the assumptions of fracture mechanics should be valid for describing the problem. Assuming that the blunt defect is a sharp crack will give conservative answers, but they may not be realistic. Some accounting must be made of the life spent initiating and growing a crack from the pore to a fracture mechanics size flaw. This initial period of growing a crack can be a significant part of the total life, especially for high cycle fatigue.

The general finding in the literature is that porosity does not behave like planar weld defects, such as lack of fusion, which are more clearly crack-like. (See, for example, References 2 and 8.)

### 2.2.2 Pore Geometry and Interaction

Porosity, though generally spherical in shape, can assume many shapes and configurations. These include elongated pores, rows of single pores or collinear pores, and pore clusters. Determining the effects of various sizes and shapes of pores is an important factor affecting the structural integrity of weldments. Unfortunately, almost no work reported

in the literature has dealt directly with the mechanisms of crack growth from potentially interacting voids. Instead, researchers have concentrated on correlating total fatigue lives with parameters describing the weld porosity. Examples are percent of porosity, reduction in area, and maximum pore size. From these indirect measurements one may be able to extract some of the rules governing the interaction of pores.

### 2.2.3 Residual Stresses

Residual stresses have been shown to significantly decrease the fatigue life of welds [8-10]. Compared to welds not containing residual stresses, tensile residual stresses can decrease the life, while compressive residual stresses can increase the life. Measurements in HY-80 butt welds have revealed longitudinal and transverse residual stresses locally as high as the yield strength [8]. Similar results have been found for mild steel butt welds [11]. Residual stress magnitudes and distributions can vary greatly [8,10]. Generally, tensile stresses are seen at the surfaces and compressive stresses at mid-thicknesses. Because of this variation, the initiation and propagation of a fatigue crack may depend on its position in the weld--i.e., on its position in the residual stress field.

### 2.2.4 Threshold Crack Growth Behavior

Below some arbitrary crack growth rate, from an engineering viewpoint, a crack is not of concern because it does not threaten the integrity of the structure in a reasonable amount of time. Although there is some debate concerning the determination of threshold stress intensities, the concept is an important one for the present study.

It has been noted that under variable amplitude loading, threshold behavior may not be as significant as under constant amplitude loading  $^{[12]}$ . This is because there will probably be some large loads which cause the small crack to grow; and as it does, more and more of the load spectrum will produce stress intensities above the threshold values.

### 2.2.5 Crack Retardation

Under variable amplitude loading similar to actual service conditions, linear elastic fracture mechanics methods have been shown to give overly conservative crack growth predictions under actual ship load histories when load interactions are not accounted for [12]. Large loads, such as bottom slamming, superimposed on smaller loads, such as low frequency wave induced stresses, result in crack growth retardation, which slow crack growth below rates that would be expected by additive linear cumulative damage.

### 3. \_SCOPE

The objective of this study was to research and define the parameters which affect the fatigue performance of marine weldments containing porosity. A model which accounts for the defined parameters was developed and exercised to study the sensitivity of fatigue life upon these factors. The model uses both low cycle fatigue concepts and fracture mechanics techniques to predict fatigue crack initiation and subsequent growth. It is important to emphasize that all of the predictions performed during this study were for weldments with the reinforcement removed. Weldments with reinforcement left intact will generally fail at the weld toe which proves to be a much more severe defect than internal porosity [1-5].

The developed model was used to predict fatigue lives of tests performed on a limited number of weld specimens containing internal porosity as a calibration exercise. The predicted lives were generally within a factor of two of the actual lives.

Four types of porosity were examined using the predictive model: uniform porosity, a single pore, co-linear porosity and cluster porosity. Fatigue life predictions are made for each of the porosity types using different plate thicknesses, residual stresses, pore sizes, and loading. For constant amplitude loading, three stress ratios are used. A variable amplitude history based upon SL-7 stress data was developed and applied in the model for all four types of porosity. The

material used for all the predictions is EH36. Because the fatigue and crack growth properties of a wide class of steels do not differ significantly from this material, the trends developed are probably applicable to many ship steels.

### 4. LITERATURE SURVEY

The work in the literature review was directed at definition of the problem, identification of factors controlling fatigue life and identification of available life prediction concepts and approaches to deal with porosity. Areas of emphasis were: stress analysis and stress-intensity solutions for volumetric stress raisers; weld induced residual stress fields; nondestructive inspection sensitivity and threshold in the laboratory and in field applications; materials, da/dN, and  $K_{\rm IC}$  for marine materials, particularly those with porosity problems; and analysis methods used to assess porosity effects on integrity.

### 4.1. Stress Analysis and Stress-Intensity Solutions for Volumetric Stress Raisers

### 4.1.1. Stress Analysis of Cavities

Sternberg  $^{[13]}$  and Savin  $^{[14]}$  have made literature surveys on theoretical stress concentration factors for cavities and holes. These references list the papers related to three-dimensional stress concentrations around spherical, spheroidal and ellipsoidal cavities in an infinite or finite elastic medium. The mutual effect of two or more spherical cavities in an infinite body and the interference between a spherical cavity and external boundary are also included in these references. Tsuchida and Nakahara  $^{[15]}$  studied a three dimensional stress concentration around a spherical cavity in a semi-infinite elastic body. Mokarov  $^{[16]}$  experimentally determined the stress distribution around a chain consisting of three spherical pores and a chain consisting of two different pores.

Lundin $^{[17]}$  described the primary types of porosity that may be of concern in welding as follows: (1) uniformly scattered (distributed)

porosity; (2) cluster (localized) porosity; (3) linear (aligned) porosity; (4) wormhole (elongated) porosity. (Porosity in weld metals is generally spherical or wormshaped. Elongated spherical porosity is rarely found in the weld metal.) Masubuchi  $^{[18]}$  has shown that stress concentration factors around porosity (under uniaxial loading) are generally below  $K_t = 4.0$ . Stress concentration factors around porosity are generally low. A qualitative discussion of stress fields near cavities is presented in Section 6 titled "Ellipsoidal Cavities".

### 4.1.2. Stress Intensity Factor for Volumetric Stress Raiser

Using a superposition method,  $Krstic^{[19]}$  obtained a stress intensity factor solution for an annular flaw emanating from the surface of a spherical cavity. Stress intensity factor handbooks  $^{[20,21]}$  contain three-dimensional solutions for circular and elliptical cracks in a solid.

### 4.2. Weld-Induced Residual Stress Fields

In Chapter 6 of Reference 22, Masubuchi has a comprehensive discussion of the magnitude and distribution of residual stresses in steel, aluminum alloys, and titanium alloys weldments. Local residual stresses at the surface of pores are not reported in the literature.

The fatigue severity of porosity relative to other weld discontinuities such as weld toe or ripple depends on both the stress concentration factors and residual stresses. Porosity which is located in zones of high tensile residual stresses might be the critical sites for fatigue failure. Babev<sup>[23]</sup> has found that the dimensions and distributions of porosity had little influence on the fatigue resistance of welds if it is located in a high residual tensile stress field.

# 4.3. Nondestructive Inspection Sensitivity and Threshold in the Laboratory and in Field Applications

 $\text{Barsom}^{\left[24\right]}$  has found that the probability of detecting small discontinuities is remote. Porosity might obscure other defects. For

example, planar defects may be embedded in cluster porosity and can not be detected using nondestructive methods.

# 4.4. Fatigue Crack Growth Data, Fracture Toughness, and Strain-Controlled Fatigue Behavior for Marine Materials (Particularly Those With Porosity Problems)

Masubuchi [22,25] has extensively reviewed the materials used for marine engineering. Marine welded structures are primarily made of steels, aluminum alloys, and titanium alloys. The steels include carbon steels, high strength low alloy steels, quenched-and-tempered steels, and maraging steels. Aluminum alloys in the 5xxx series and the 7xxx series are used extensively in marine applications. Among the titanium alloys, pure titanium and the Ti-6Al-4V alloy have been most commonly used. Although there are many causes of porosity in fusion welds, aluminum alloys and titanium alloys are more active than steels and thus prone to weld porosity.

### 4.4.1 Fatigue Crack Growth Data

Hudson and Seward [26,27] have compiled a list of sources of fracture toughness and fatigue crack growth data for alloys. This list covers many marine metallic materials. Most of the fatigue crack growth data is for the base metal. There is very little data available for weld metals and heat affected-zone (HAZs). Maddox [28] has conducted tests on a variety of structural C-Mn steels base-metals, weld-metals, and HAZs. The test results show that the rates of fatigue crack growth in weld metals and HAZs are equal or less than that in the base metal. Therefore, the upper scatter band of fatigue crack growth rates for base metals can be used to obtain conservative engineering estimates of the fatigue crack growth rates in base metals, weld metals, and HAZs. Barsom[29] has suggested upper scatter band equations for martensitic steels, ferritic-pearlitic steels, and austenitic steels.

### 4.4.2. Fracture Toughness

In general, there are four types of fracture toughness tests used for marine welded structures  $^{[30]}$ : (1) the Charpy impact tests; (2) the Drop Weight tests (DWT), or the closely related Dynamic Tear Test; (3) fracture mechanics tests to measure critical stress intensity factors (K\_C or K\_IC) or critical values of the J-integral (J\_C or J\_IC); (4) the Crack-Tip-Opening Displacement (CTOD or COD) test. Masubuchi, et al.  $^{[31]}$  have done a literature survey on the notch toughness of weld metals and the HAZs, evaluated primarily by the Charpy V-notch impact test. Ship Structure Committee Reports 248  $^{[32]}$  and 276  $^{[33]}$  present fracture toughness characterization of ship steels and weldments using Charpy impact test, DWT test, and explosion structural tests. References  $^{[26,27]}$  list fracture toughness for many of the marine metallic materials. Lawrence, et al.  $^{[34]}$  studied the effects of porosity on the fracture toughness of three aluminum alloy weldments using DWT energy and J integral.

### 4.4.3. Strain-Controlled Fatigue Behavior

Very few strain-controlled fatigue properties are available for marine materials. References  $^{\left[35,36\right]}$  provide several cyclic fatigue properties for the base metals, weld metals, and HAZs of various steels and aluminum alloys.

### 4.5. Analysis Methods Used to Assess the Effects of Porosity on Structure Integrity

British Standards institute Document PD6493:1980<sup>[37]</sup> provides guidance on some methods for the derivation of acceptance levels (fitness for service) for defects in fusion welded joints. In the section below, the analysis methods used to assess the effect of porosity on the fatigue performance of weldments will be discussed.

### 4.5.1 Previously Used Methods

### 4.5.1.1. Harrison's "Quality Bands" Method

Harrison<sup>[1]</sup> presented a fitness-for-service evaluation of porosity as shown in Figure 1. The levels shown for quality bands denoted as V, W, X, Y, Z and corresponding to 0, 3, 8, 20 and 20+ percent porosity were drawn based on the available data. Figure 1 also shows the comparison of quality band method with fatigue test results. This method generally gives conservative and lower-bound fatigue resistance estimates for weldments with porosity.

### 4.5.1.2. Hirt and Fisher's LEFM Analysis

Hirt and Fisher<sup>[38]</sup> have studied the influence of porosity on the fatigue behavior of longitudinal web-to-flange welds by assuming the pores to be circular penny-shaped cracks. Linear elastic fracture mechanics was used to calculate the fatigue crack propagation life. This approach may be very conservative because the pores are generally rounded.

### 4.5.2. An Analysis Based on Total Fatigue Life - A Proposal

The most serious deficiency of the method of Hirt and Fisher is the neglect of the period of life devoted to fatigue crack initiation and early growth. A more accurate assessment of the effects of porosity on the fatigue life of marine structures could be obtained by adding estimates of fatigue crack initiation life to the fatigue propagation life using methods such as those of Lawrence, et al.  $^{\left[39\right]}$  and Reemsnyder  $^{\left[40\right]}$ . Both of these methods provide estimates of the fatigue crack initiation life and consider the important effects of mean and residual stresses. While LEFM provides good estimates of long crack growth, methods developed by Leis  $^{\left[41\right]}$  could be used to improve the accuracy of fatigue crack propagation life estimates for the portion of the fatigue crack propagation life in which the dominant crack is located within the inelastic stress field of the notch (pore).

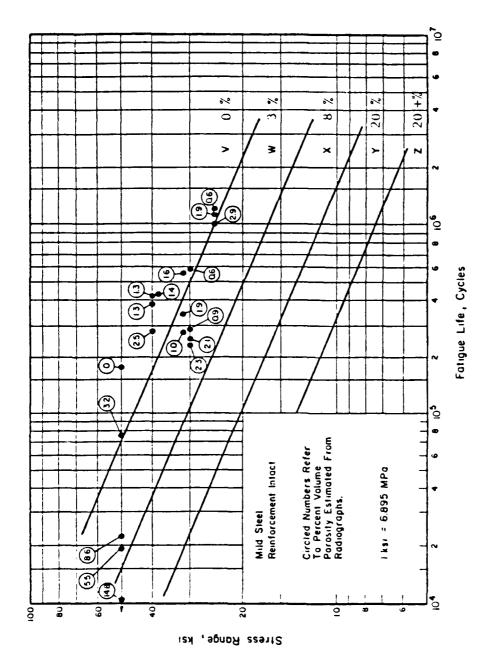


FIGURE 1. COMPARISON OF FATIGUE TEST RESULT WITH QUALITY BAND APPROACH FOR POROSITY

### 5. ANALYTICAL MODELING BACKGROUND

The model used to predict the fatigue lives of weldments used during this study consists of two parts; the crack initiation life,  $N_i$ , in cycles, and the crack propagation life,  $N_p$ , in cycles. The sum of these two components is the total life,  $N_t$ ,

$$N_i + N_p = N_t \tag{1}$$

The crack initiation life is estimated using low cycle fatigue concepts and the crack propagation life is estimated using linear elastic fracture mechanics concepts. The intent of this section is to provide the low cycle fatigue and fracture mechanics background used in the development of the predictive model. In Section 7, titled <u>Analytical Program</u>, these concepts will be applied to single pores, co-linear porosity, uniform porosity, and pore clusters.

### 5.1 Initiation Life Model

Fatigue cracks generally initiate at a geometrical discontinuity such as a notch or pore. These act as stress concentrations, raising the stress in the region of the notch to levels above the nominal stresses. The material at the notch root may deform plastically while the rest of the component remains essentially elastic. Subjecting the region to cyclic loading resulting in plastic deformation will eventually result in a fatigue crack.

### 5.1.1 Notch Analysis

Determining the stresses and strains in the notch region after the onset of local plasticity requires a notch analysis technique. In the elastic range, the notch stress can be calculated using the elastic stress concentration factor,  $K_{\tt t}$ . The  $K_{\tt t}$  value is simply a conversion

factor between the maximum principal notch stress,  $\sigma$ , and remote stress, S,

$$\sigma = K_{t} S , \qquad (2)$$

and is determined using elasticity theory or by finite element analysis. After the notch region material deforms plastically, however, the elastic stress concentration factor no longer applies as a direct conversion factor. The stress will rise at a lesser rate and the strain at a greater rate than during elastic deformation where both stress and strain rates were equal. Neuber's rule  $^{\left[42\right]}$  is used to estimate the local stresses and strains in this situation. Nueber's rule states that the elastic stress concentration,  $K_{t}$ , will remain equal to the geometric mean of the instantaneous stress and strain concentration factors,  $K_{\sigma}$  and  $K_{\varepsilon}$ , respectively,

$$K_{t} = \left(K_{\sigma} K_{\epsilon}\right) \tag{3}$$

Rewriting this relation in terms of stress and strain ranges as

$$K_{t} = \left(\frac{\Delta \sigma \Delta \epsilon}{\Delta S \Delta e}\right)^{1/2}$$

where  $\Delta S$  is the nominal stress range, and  $\Delta e$  is the nominal strain range, and recalling that

$$\Delta e = \Delta S / E$$
 (4)

where E is the elastic modulus, Neuber's rule may be written for nominally elastic response as

$$\frac{\Delta S^2 K_t^2}{E} = \Delta \sigma \Delta \epsilon$$

This expression relates the local stress-strain response at the notch root to the nominal stress and elastic stress concentration factor. Furthermore, representing the stress-strain response of the material with power law hardening constants,

$$\Delta \epsilon = \frac{\Delta \sigma}{E} + \left(\frac{\Delta \sigma}{K}\right)^{1/n} \tag{5}$$

where K is the strength coefficient, and n is the strain hardening exponent, the relation can be written with  $\Delta \sigma$  as the only unknown,

$$\frac{\Delta S^2}{E} \qquad K_t^2 = \Delta \sigma \left( \frac{\Delta \sigma}{E} + \left( \frac{\Delta \sigma}{K} \right)^{1/n} \right)$$

Solving for  $\Delta \sigma$  is accomplished using an iterative technique such as Newton's method.

### 5.1.2 Fatigue Notch Factor

In fatigue testing, it is generally observed that the actual lives of notched components are somewhat longer than would be expected for the notch root stress calculated using the elastic stress concentration factor,  $K_t$ . That is, notches have a less detrimental effect on fatigue life than would be predicted. This effect is dependent upon both defect size and material. To account for this difference, a fatigue notch factor,  $K_f$ , is often used in place of  $K_t$  for fatigue life predictions. The fatigue notch factor is defined as

$$K_f = \frac{\sigma_{\text{unnotched at a finite life (e.g. 10}^7)}}{\sigma_{\text{notched}}}$$
 (6)

The value of  $K_f$  for a given notch geometry and material can be determined experimentally or by the use of analytical relations. A commonly used fatigue notch factor relation is Peterson's equation [43],

$$K_{f} = 1 + \left(\frac{K_{t} - 1}{1 + a/r}\right) , \qquad (7)$$

where a is a material constant dependent on strength and ductility and r is the notch tip radius. The material constant a can be approximated for ferrous-based wrought metals by an equation fitted to Peterson's data,

$$a = \left(\frac{300}{S_u}\right)^{1.8} \times 10^{-3} \text{ in.}$$
 (8)

where  $S_{\bf u}$  is the ultimate strength in ksi units. Peterson's equation indicates that small notches are least sensitive in fatigue, and that ductile materials are less sensitive to notches in fatigue than strong materials.

### 5.1.3 Notch Strains and Low Cycle Fatigue

Using Nueber's rule for notch root stress-strain behavior along with Peterson's equation for the fatigue notch factor, it is possible to estimate the stress-strain response of the notch root material subjected to fatigue loading. It still remains to relate these local stresses and strains to actual fatigue life data. Because the plastically deformed notch root material is constrained by the surrounding elastic material, the notch root is nearly in a strain-control condition. The notch root material is essentially cycled between strain limits analogous to strain-control, low cycle fatigue testing. The assumption, therefore, is that strain-life fatigue data obtained using unnotched, low cycle fatigue specimens can be used to predict the cycles to crack initiation,  $N_{\rm i}$ , at a

notch root. Low cycle fatigue strain-life data is often represented by the Coffin-Manson equation with Morrow's mean stress correction,

$$\frac{\Delta \epsilon}{2} = \epsilon_f' (2N_f)^c + \left(\frac{\sigma_f' - \sigma_m}{E}\right) (2N_f)^b$$
 (9)

where  $\Delta E/2$  is the strain amplitude,  $E_f$  is the fatigue ductility coefficient,  $\sigma_f$  is of the fatigue strength coefficient,  $\sigma_m$  is the mean stress,  $2N_f$  is the reversals to failure,  $N_f$  is the cycles to failure, c is the fatigue ductility exponent, and b is the fatigue strength exponent. By relating the strain calculated at the notch root to the strain-life data, the number of cycles to initiate a fatigue crack at the notch can be estimated. This is the basis of the initiation life predictions. The strain-life data parameters,  $E_f$ ,  $\sigma_f$ , c, and b, are obtained either by low cycle fatigue testing or by using estimates. [44]

### 5.2. Propagation Life Model

### 5.2.1. Fatigue Crack Growth Rate

Paris and  $Erdogan^{[45]}$  have shown that fatigue crack growth rates are dependent upon the stress intensity associated with the fatigue crack tip. The power-law relationship is of the form

$$\frac{da}{dN} = A \Delta K^{m} \tag{10}$$

where da/dN is the fatigue crack growth rate,  $\Delta K$  is the stress intensity factor range, and A and m are material constants dependent upon environment, stress ratio, temperature, and frequency. This relationship is considered valid above an experimentally determined threshold stress intensity value. Below the threshold value, fatigue cracks grow so

slowly as to be of no practical consequence. The growth rate expression used throughout this study has a correction factor to account for mean stress effects,

$$\frac{da}{dN} = \frac{A \Delta K^{m}}{1-R}$$

where R is the stress ratio,

$$R = S_{min}/S_{max}$$

### 5.2.2. Stress Intensity Factor

The general relationship for the stress intensity factor range is written as

$$\Delta K = Y \Delta S (\pi a)^{1/2} \tag{12}$$

where Y is a geometry dependent factor,  $\Delta S$  is the stress range, and a is the crack length. The geometry factor Y is actually composed of a number of separate multplicative geometry factors which account for the shape of the crack, the thickness of the component or specimen, and the position of the crack within the body. The value Y is written as

$$Y = \frac{M_s M_t M_k}{\Phi_0}$$
 (13)

where  ${\rm M}_{\rm S}$  accounts for the free front surface,  ${\rm M}_{\rm t}$  accounts for the finite plate thickness,  ${\rm M}_{\rm k}$  accounts for the nonuniform stress gradient due to the stress concentration of the geometric discontinuity, and  $\Phi_0$  accounts for the crack shape.

The  $\rm M_{\rm S}$  factor, which accounts for the front free surface, is expressed by the relation  $^{\left[46\right]}$ 

$$M_s = 1.0 - 0.12(1 - a/2c)^2$$
 (14)

where a/c is the ratio of the minor and major ellipse axes. The majority of cracks examined in this study, however, are embedded in the material, so the free surface correction is equal to unity.

The  $M_t$  factor, which accounts for the finite plate thickness, is found in stress intensity handbooks such as [20,21]. The  $M_k$  factor requires a brief explanation. The need for such a factor arises because the stress,  $\sigma$ , near a discontinuity is greater than the remotely applied stress, S, used to calculate  $\Delta K$ . A crack tip growing through the stress gradient is therefore subjected to higher stresses which result in a greater stress intensity factor range,  $\Delta K$ . Not accounting for this increase in stress intensity would lead to unconservative predicted growth rates near the discontinuity. The discrepancy in total life would be greatest for large notches because the stress gradient is sustained in proportion to the absolute notch size. The subject of stress intensity factors in stress gradients is examined by Albrecht and Yamada [47]. The method presented in Reference 47 is used to calculate  $M_k$  in the present study.

The crack shape correction factor,  $\Phi_0$ , is expressed by the integral

$$\Phi_0 = \int_0^{\pi/2} \left[1 - (1 - a^2/c^2) \sin^2 \Phi\right]^{1/2} d\Phi$$
 (15)

where a is the length of minor axis of ellipse and c is the length of the major axis.

### 6. STRESS FIELDS NEAR INTERNAL CAVITIES

Porosity is defined as cavity type discontinuities (voids) formed by gas entrapment during solidification. The shape of the void is dependent on the relative rates of solidification of the weld metal and the nucleation of the entrapped gas. The resultant stress field surrounding the pore depends upon the pore shape and the loading.

### 6.1. Ellipsoidal Cavities

The shape of porosity can be generalized for analytical purposes as an ellipsoid. The coordinate system defining the cavity is shown in Figure 2. Pore shapes can range from an oblate ellipsoid (a=b=1) to a sphere (a=b=c=1) to a prolate ellipsoid (b=c=1) or any shape in between, as shown in Figure 3. The elastic solution for the stress field around a triaxial ellipsoidal cavity in an infinite medium has been found by Sadowsky and Sternberg [48]. The stress in the plots in Figure 3,  $\sigma_{\rm Z}$ , is the local stress resulting from an applied uniaxial stress, S<sub>7</sub>, of unity.

Some general characteristics of the stress fields are worth noting. Subject to a uniaxially applied stress of  $S_Z$ , the maximum stress concentration will always occur at the minor axis of the x-y plane ellipse, point B. The stress  $\sigma_Z$ , therefore, is plotted relative to point B along the y axis. In the limiting cases, when a=b=1 and c approaches 0, the stress  $\sigma_Z$  tends toward infinity, representing the case of an embedded penny-shaped crack. As c approaches infinity,  $\sigma_Z$  tends toward the remote stress,  $S_Z$ . When b=c=1, and a also equals 1, the solution is that for a sphere. As a approaches infinity, the solution coincides with that of a hole in a plate with a stress concentration of 3.

These solutions are for cavities in an infinite medium. In application to weld porosity, they are valid if the size of the cavity is small in relation to the dimensions of the weldment.

### 6.2. Spherical Cavities in a Semi-Infinite Medium

The elastic solution for the stress field near a spherical cavity in a semi-infinite medium has been found by Tsuchida and Nakahara  $^{[15]}$ . Figure 4 shows the effect of increasing stress concentration as the distance between the surface and the pore decrease. The plot also shows that the presence of the surface has little effect on the stress field

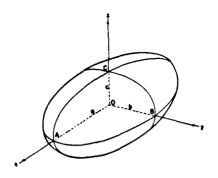


FIGURE 2. ELLIPSOIDAL CAVITY AND CARTESIAN CO-ORDINATE SYSTEM

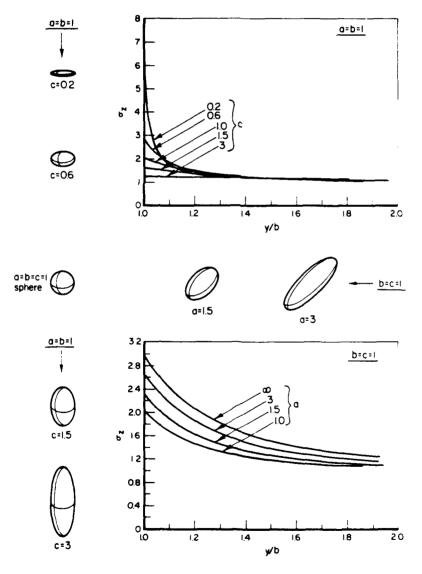


FIGURE 3. LOCAL STRESS,  $\sigma_Z$ , ALONG Y AXIS, FOR VARIOUS ELLIPSOIDAL CAVITIES SUBJECTED TO NOMINAL STRESS,  $S_Z$ , OF UNITY

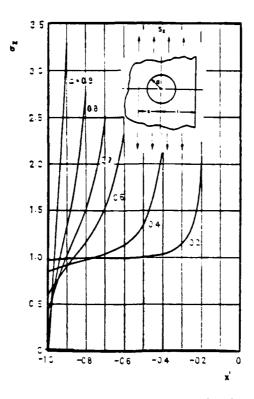


FIGURE 4. LOCAL STRESS,  $\sigma_Z$ , ALONG X' AXIS, FOR SPHERICAL CAVITY NEAR A SURFACE, SUBJECTED TO NOMINAL STRESS,  $S_Z$ , OF UNITY

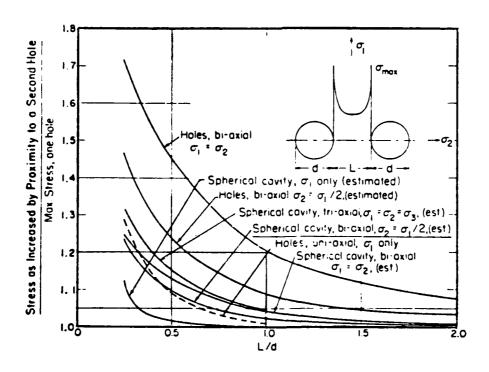


FIGURE 5. INTERACTION EFFECT OF TWO HOLES OR CAVITIES IN AN INFINITE PLATE OR BODY

when the ratio of the pore radius to the distance between pore center and surface is less than 0.4.

### 6.3. Cavity Interaction

The problem of cavity interaction is complex and correspondingly there is little information available on the topic. Sadowsky and Sternberg  $^{[48]}$  examined the problem and solved two specific cavity spacings for triaxial loading. Peterson  $^{[49]}$  took these results and made approximations for the uniaxial case. The results are presented in Figure 5 along with solutions for holes. During the present study, cavity interaction was assumed only for the case of cluster porosity where pores are expected to be in close proximity to each other. All other pores were assumed to be non-interacting. Markarov  $^{[16]}$  has demonstrated through photoelastic techniques that cavities separated by two pore diameters do not effect the stress distribution of the other.

### 7. ANALYTICAL PROGRAM

### 7.1. Application of Initiation-Propagation Model to Porosity

### 7.1.1 Initiation Life

Volumetric discontinuities such as pores act as relatively mild stress concentrations because of their rounded asperities. A spherical cavity, for instance, has a stress concentration factor of only 2.05 (with Poisson's ratio of 0.3). The low stress concentration suggests that a fatigue crack would take a large number of stress cycles to initiate. For smaller pores more cycles would be needed because of the fatigue notch size effect,  $K_{\mathbf{f}}$ . Larger pores would be expected to initiate cracks sooner.

### 7.1.2 Propagation Life

When a crack does form, it initially has a high stress intensity factor range,  $\Delta K$ , while growing through the pore stress gradient. The stress gradient, however, decays rapidly as is characteristic of volumetric defects. The larger the pore size, the longer the distance that the crack is subjected to the higher stress because the gradient is sustained in proportion to the absolute pore size. The crack shape is assumed to remain circular while it propagates. A circular crack shape is the most energetically stable planar flaw configuration for Mode I crack growth. Considering Equation 13,  $\Phi_0$  for a circular crack is 1.57 whereas  $\Phi_0$  for an elliptical crack with a small a/c aspect ratio is nearly 1.0. This means that a circular crack will have only 0.6 times the stress intensity factor range,  $\Delta K$ , than an elliptical crack with a small aspect ratio and an equal crack front (a) dimension.

A plasticity crack length correction factor was not used in the crack growth calculations. The generally low stresses (nominally elastic) used in this study results in a small plastic zone size at the crack tip. The confined yield zone assumption means that LEFM is valid for most of the propagation calculation.

### 7.1.3 Initial Crack Size

The initial crack size used in the propagation estimates was taken as 0.05 times the pore diameter. This assumption starts the crack at the same distance relative to the stress gradient in all cases. The initial crack length is considered to be beyond the region were anomalous crack growth behavior when analyzed in terms of LEFM occurs. Smith and Miller  $^{[50]}$  found that the transition length between anomalous behavior and that governed by LEFM to be 0.065 times the diameter for a circular hole. This distance would be expected to be somewhat less for a three-dimensional flow such as a pore.

### 7.1.4 Failure Criteria

The failure criteria for all cases is through thickness cracking.

# 7.2. Viability of the Fatigue Life Model

The literature was searched for fatigue tests on weldments containing porosity with sufficient documentation to apply the predictive model. The most useful type of documentation was fractographs of the surfaces which clearly showed the sizes, shapes, and positional relationships of the porosity. Only two test programs  $\begin{bmatrix} 6,51 \end{bmatrix}$  were found which included such fractographs. A total of eight fatigue tests were found to which the model could be applied. Neither of these test programs, however, included material property data for the weld metal. Both test series used E70 weld metal in a gas-metal-arc welding process. The method for introducing porosity into the weld metal was interruption of the shielding gas flow in both studies.

Because no fatigue material property data was available for E70 weld metal, E60 S-3 (2 pass) weld metal  $^{\left[36\right]}$  properties were used as the baseline data. The mechanical properties of E60 S-3 (2 pass) weld metal is shown in Table 1 and Figures 6 and 7.

Leis, et al.<sup>[6]</sup> performed axial fatigue tests on pipe wall segments with girth welds in A106B steel. The weld reinforcement was left intact, but the weld toe was ground to a large radius to cause fatigue crack initiation from the internal flaws. Three tests contained sufficient porosity that allowed application of the model. The fractographs of these specimens are shown in Figure 8(a-c). The porosity clusters are ellipsoidal in shape and include individual pores of approximately 0.02 inches in diameter. Within the cluster area, the percent porosity is approximately forty percent by area.

Ekstrom and Munse [51] performed fatigue tests on a double V butt weld geometry. In this test program, the reinforcement was completely removed to cause internal crack initiation. Five tests included welds with severe porosity. The fracture surfaces for these test pieces are shown in Figure 8(d-h).

TABLE 1. MECHANICAL PROPERTIES OF E60 S-3(2P) WELD METAL

E	27400 ksi	188923 MPa
S	59 ksi	408 MPa
S <sub></sub>	84 ksi	579 MPa
% RA	60.7	60.7
$\sigma_{_{\mathbf{f}}}$	126 ksi	869 MPa
€,	0.933	0.933
$\sigma_{u}^{i}$	53 ksi	373 MPa
K,	179 ksi	1234 MPa
יח	0.197	0.197
$\sigma_{\mathrm{f}}^{i}$	149 ksi	1027 MPa
b <sup>'</sup>	-0.09	-0.09
€¦	0.602	0.602
c	-0.567	-0.567
Α	2.69×10 <sup>-12</sup>	3.95X10 <sup>-14</sup>
m	5.8	5.8
	Sys RA  Sys R	S <sub>y</sub> 59 ksi S <sub>y</sub> 84 ksi % RA 60.7 σ <sub>f</sub> 126 ksi ε <sub>f</sub> 0.933  σ' <sub>y</sub> 53 ksi κ' 179 ksi n' 0.197 σ' <sub>f</sub> 149 ksi b -0.09 ε' <sub>f</sub> 0.602 c -0.567

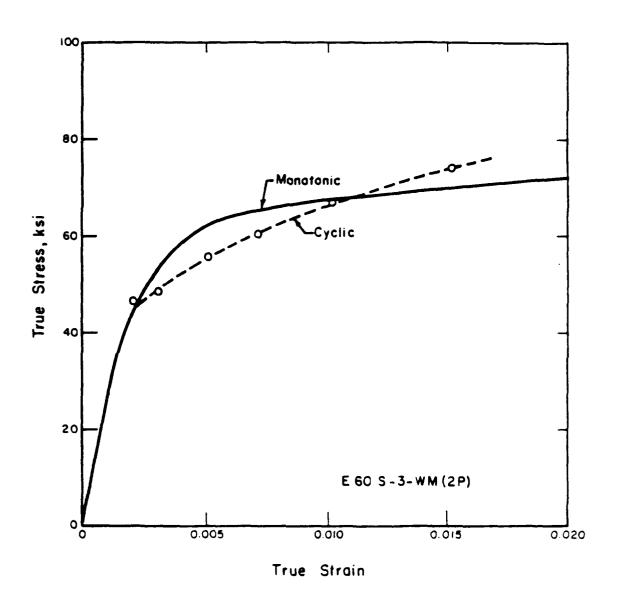


FIGURE 6. MONOTONIC AND CYCLIC STRESS-STRAIN RESPONSE FOR E60 S-3 WELD METAL (2 PASS)

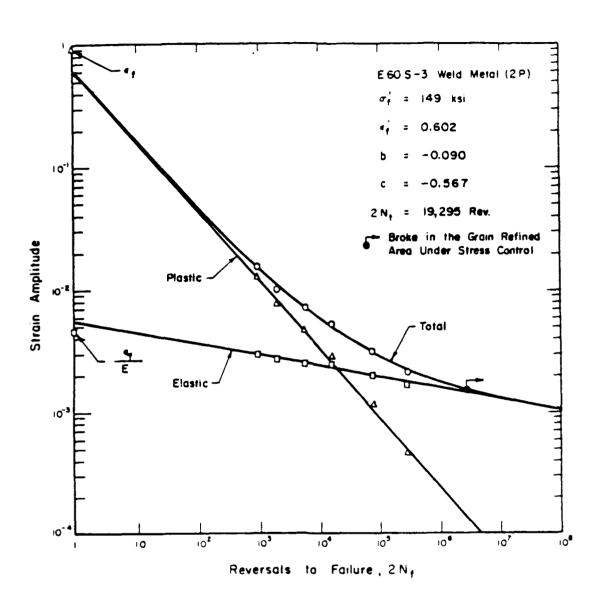


FIGURE 7. STRAIN-LIFE DATA FOR E60 S-3 WELD METAL



(a) CPN-2 Stress Range 27.5 ksi. Life - 2,115,600

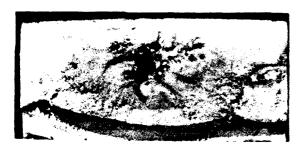


(b) CPN-4 Stress Range 33 ksi, Life - 54,600



(c) CPN-5 Stress Range 27.5 ksi, Life - 334,100

FIGURE 8. FRACTURE SURFACES OF WELDS WITH CLUSTERS OF POROSITY



(d) PS 5-1 Stress Range 34 ksi Life - 713,300



(e) PS 5-2 Stress Range 34 ksi Life - 325,500



(f) PS 5-3 Stress Range 44 ksi Life - 80,300



(g) PS 5-4 Stress Range 29 ksi Life - 633,000



(h) PS 5-5 Stress Range 27 ksi Life - 1,024,900

FIGURE 8. FRACTURE SURFACES OF WELDS WITH CLUSTERS OF POROSITY (Continued)

Fatigue life predictions were made for all eight tests using the model described in Section 7.3.6. All the individual pores were assumed to be spherical so an elastic stress concentration factor,  $K_{+}$ , of 2.05 was applied. In those cases were interaction was assumed an additional factor of 1.12 was applied. Table 2 lists the experimental test results and the fatigue predictions for each test. For each test, the following predictions are presented: predicted fatigue life at the specified test stress range; predicted stress range for the specified fatigue life; predicted fatigue life for specified test stress range treating the porosity cluster as a gross ellipsoidal cavity with dimensions a, b, and c; and fatigue life predictions using only the reduced cross sectional area without assuming a stress concentration. The results show that treating the pore cluster as a gross ellipsoidal cavity is somewhat conservative while considering the flaw as merely a reduction in cross sectional area is very unconservative. Applying the model for cluster porosity resulted in good estimate for fatigue life and, when viewed in terms of stress, even better estimates. The absolute magnitude of the predictions are not as important as the trends because of the uncertainty in material properties. Figure 9(a) shows the comparison between experimental and predicted fatigue lives and Figure 9(b) shows the comparison between the experimental and predicted stress ranges for the test life.

The predicted lives are dominated by the crack initiation period. This is due mainly to the size of the defects with respect to the cross sectional area of the specimen. The initiation life is considered to be the number of cycles until the crack begins growing radially away from the defect cluster. This includes the period of crack coalescence between the pores. After the cracks between the pores coalesce, the material at the outer portion of the periphery pores are assumed to initiate a crack and grow toward the surface. At this point the net cross sectional area is greatly decreased and the resultant higher stresses propagate the crack rapidly until failure.

These predictions are based on a limited sample of weldments and therefore can not be considered conclusive evidence that the predictive model is viable or not. It should be noted, however, that assuming an

TABLE 2. FATIGUE TEST RESULTS AND PREDICTIONS OF WELDS CONTAINING POROSITY

Specimen	Nominal Stress Stress Range, ksi Ratio	Strass Ratio	Area Percent Porosity	Gross Flaw Dim.	Gross Actual Fatigue Flaw Dim. Life, cycles	•	Predicted F	Predicted Fatigue Life, cycles Ni Nt	cycles Nt	Predicted Stress Range, ksi
CPN-4	32.7	0.1	ю. Э	a=0.67 b=0.075 c=0.038	54,600	Cluster Method: Gross Flaw: Percent Area:	135,083 1,271	89 89	135,151 1,339 3.0e8	36.6
CPN-2	27.2	0.1	න භ	a=0.80 b=0.063 c=0.032	2,115,600	Cluster Method: Gross Flaw: Percent Area:	771,973 3,106	319 319	772,292 3,425 2.3e9	24.9
CPN-5	27.2	0.1	11.8	a=0.75 b=0.12 c=0.032	334,100	Cluster Method: Gross Flaw: Percent Area:	463,788 145	17	463,805 162 6.7e9	28.6
PS5-3	0.44	0.222	₩.	a=0.34 b=0.13 c=0.078	90,300	Cluster Method: Gross Flaw: Percent Area:	21,540	12	21,552 1,186 1.7e7	39.2
PS6-2	34.0	-0.056	<b>4</b> . <b>8</b>	a=0.29 b=0.14 c=0.062	325,500	Cluster Method: Gross Flaw: Percent Area:	570,142 1,534	29	570,171 1,563 2.9e7	35.8
PSS-1	34.0	-0.056	2.2	a=0.27 b=0.12 c=0.12	713,300	Cluster Method: Gross Flaw: Percent Area:	717,814 30,865	394 394	718,208 31,259 3.7e7	34.1
PSS-4	29.0	0.195	3.1	a=0.43 b=0.12 c=0.093	633,000	Cluster Method: Gross Flaw: Percent Area:	444,028 8,778	119 119	444,145 8,895 7.7e7	28.2
PSS-5	27.0	0.250	4.6	a=0.39 b=0.12 c=0.062	1,024,900	Cluster Method: Gross Flaw: Percent Area:	2,177,281 2,119	142 2, 142	2,177,423 2,261 1.8e9	28.8

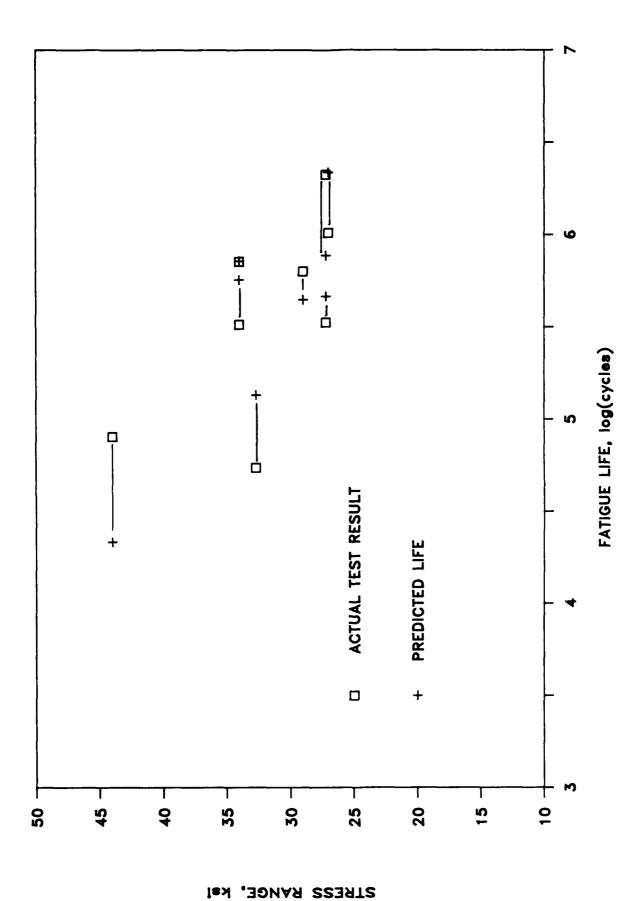


FIGURE 9(a). STRESS-LIFE PLOT SHOWING ACTUAL FATIGUE LIVES VERSUS PREDICTED FATIGUE LIVES OF WELDS CONTAINING POROSITY

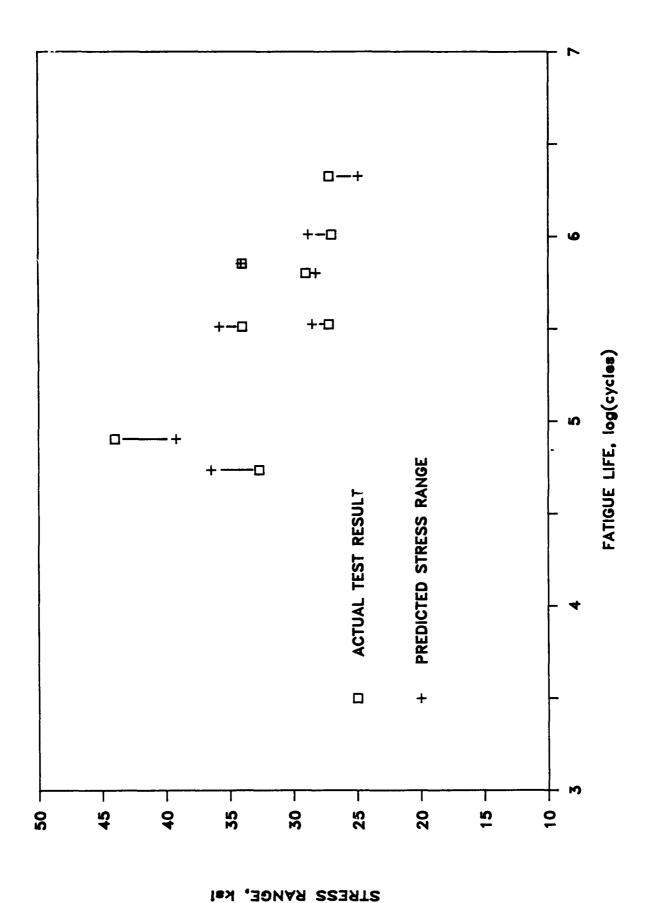


FIGURE 9(b). STRESS-LIFE PLOT SHOWING ACTUAL STRAIN RANGE VERSUS PREDICTED STRESS RANGE OF WELDS CONTAINING POROSITY

existing crack-like defect equal to the size of the cluster would lead to grossly conservative life estimates (equal to the propagation lives). The model seems to reflect the correct trends for the fatigue lives of the specimens tested. The results are even more encouraging when considering percent error in stress range predicted to yield the fatigue life of the sample. A number of uncertainties such as using approximate mechanical properties data and estimating the percent area porosity and pore sizes from photographs will certainly contribute to the scatter in the predictions. The small sample size also compounds the problem. The results are encouraging, but further testing is warranted to validate its accuracy.

### 7.3. Parametric Study

From the literature review, the parameters which have been found to influence the fatigue lives of weldments containing porosity are: weld type, material, thickness, residual stress, loading, porosity type, and pore size. In order to explore the effects of these parameters, four distinct analytical procedures are presented; one each for the four types of porosity being considered. Because of the limited amount of actual test data, the procedures rely in large part on assumptions which are considered to be consistent with the mechanisms of crack initiation and growth. The assumptions for each procedure are presented in the appropriate sections.

#### 7.3.1. Matrix of Fatigue Life Predictions

The matrix of fatigue life predictions is shown in Table 3. For the constant amplitude loading, there are 144 separate cases to be examined. Each case requires loading at four stress ranges to generate S-N curves. This represents a total of over 550 individual life predictions. All nominal fatigue loadings will be assumed to be in the elastic range. The maximum nominal load for the constant and variable amplitude loadings will be less than the yield strength of EH36, i.e. 51 ksi. Four

TABLE 3. MATRIX OF FATIGUE PREDICTIONS

Parameters	Options	
Weld type	Transverse butt we	ld
Steel	EH36	
Thickness	0.5 in., 1.0 in.	
Residual stress	+Sy, 0	
Loading:		
Constant amplitude	R = -1, 0, 0.5	
Variable amplitude	SL-7 history, 0 and 6.5 ksi mean stress bias	
	Porosi	ty Size, inch
Porosity Type	0.5-inch weld	1-inch weld

	_		Porosit	y Size, in	cn	
Porosity Type	0.	5-inch we	e1d	1-inch weld		
Uniform porosity	0.015	0.030	0.045	0.015	0.045	0.075
Single pore	0.125	0.1875	0.25	0.1875	0.25	0.30
Co-linear porosity	0.125	0.1875	0.25	0.1875	0,25	0.30
Cluster porosity	0.125	0.1875	0.30	0.1875	0.25	0.40

stress ranges; 80, 60, 40, and 20 percent of the yield strength were used to construct S-N curves.

The geometry and coordinate system used in this study is shown in Figure 10. Note that no width dimension is included on the plate. The calculations for all life estimates in the parametric analysis are based on the assumption of infinite width. This means that the size of the pore and subsequent crack will not change the nominal applied stress, S. The results can be applied to a finite geometry correcting for a decrease in net cross sectional area.

All life predictions are made for a butt weld with the reinforcement removed to model crack initiation from internal porosity. The size and number of the porosity was chosen according to Section 2.6.4: Radiographic Inspection for Porosity in the Rules for Nondestructive Inspection of Hull Welds [54]. Figures 11 and 12 show the porosity acceptance charts from this code for the thicknesses examined in this study. The code states that the maximum area percent porosity allowable in any size weld is 1.5 percent. Three porosity sizes were used. One was equal to the maximum allowable porosity size as defined in the code. The other two sizes are chosen larger than the first one.

The S-N curves presented were constructed using a smooth fit to the total lives. Cases where lives were greater than  $10^8$  are not shown on the plots. The curves terminate at the greatest predicted life less than  $10^8$ . Those predictions greater than  $10^8$  are indicated in the tables.

# 7.3.2. Material Properties

The material properties for ABS EH36 used in this study are presented in Table 4 and in Figures 13 and 14. The material is assumed to be homogeneous and isotropic. In reality, weld metal is seldom homogeneous, due to non-equilibrium cooling rates, thermal gradients, and the introduction of impurities. Also, the pressure of porosity suggests some degradation of material properties as the result of improper welding practice. However, it is beyond the scope of this study to account for any microstructural gradients due to the welding process.

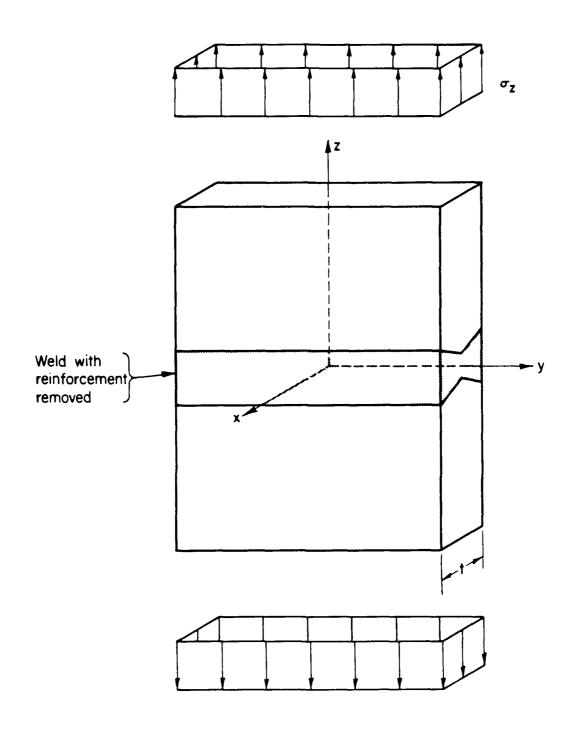
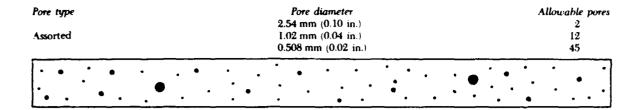
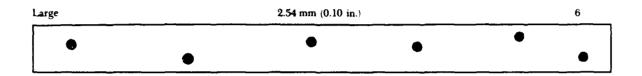
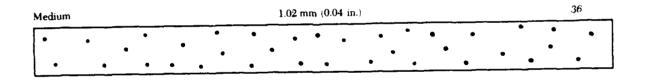


FIGURE 13. GEOMETRY AND CO-ORDINATE SYSTEM OF BUTT WELD FOR FATIGUE LIFE PREDICTIONS. THE WELD REINFORCEMENT IS REMOVED. THE WIDTH OF THE PLATE IS ASSUMED MANY TIMES THE THICKNESS OF THE WELD







Fine	0.508 mm (0.02 in.)	143
		· : : <u> </u>

FIGURE 11. CLASS A AND CLASS B POROSITY CHART FOR 0.5 INCH (12.5 MM) THICK MATERIAL

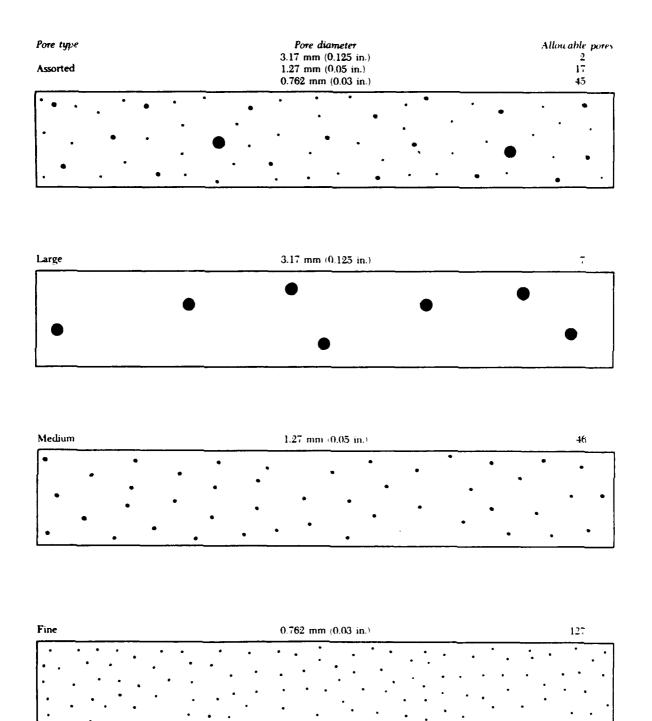


FIGURE 12. CLASS A AND CLASS B POROSITY CHART FOR 1.0 INCH (25.3 MM) THICK MATERIAL

TABLE 4. MECHANICAL PROPERTIES OF ABS EH36 STEEL

Monotonic Properties	<del></del>	_	
Young's Modulus,	E	30,700 ksi	211,677 MPa
Yield Strength (0.2%)	S <sub>u</sub>	61 ksi	421 MPa
Tensile Strength	S <sub>y</sub> S <sub>u</sub>	75 ksi	518 MPa
Reduction in Area	% RA	77.4	77.4
True Fracture Strength	$\sigma_{_{ m f}}$	186.3 ksi	1285 MPa
True Fracture Ductility	ξ	1.49	1.49
Cyclic Properties			
Cyclic Yield Strength	σ',	49 ksi	338 MPa
Cyclic Strength Coefficient	ر K	132 ksi	912 MPa
Cyclic Strain Hardening Exponent	'n	0.162	0.162
Fatigue Strength Coefficient	$\sigma_{_{\mathbf{f}}}^{_{1}}$	103 ksi	713 MPa
Fatigue Strength Exponent	p,	-0.075	-0.075
Fatigue Ductility Coefficient	€;	0.227	0.227
Fatigue Ductility Exponent	c '	-0.462	-0.462
Propagation Properties			
Crack Growth Coefficient	Α	1.76×10 <sup>-12</sup>	2.92X10 <sup>-14</sup>
Crack Growth Exponent	m	4.5	4.5

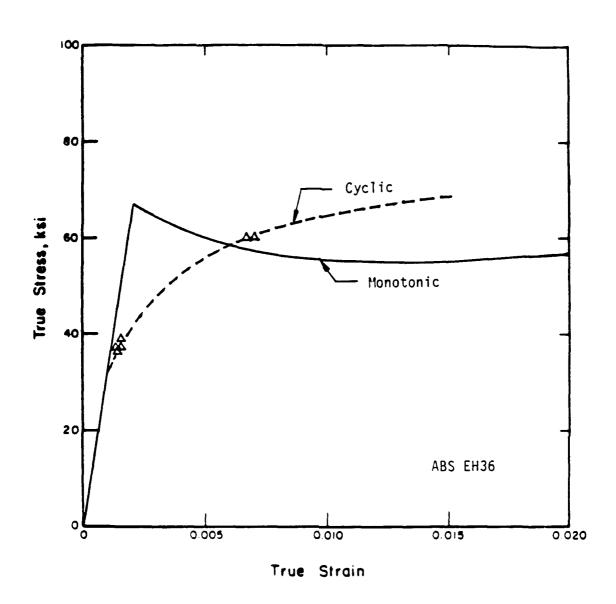


FIGURE 13. MONOTONIC AND CYCLIC STRESS-STRAIN RESPONSE FOR ABS EH36

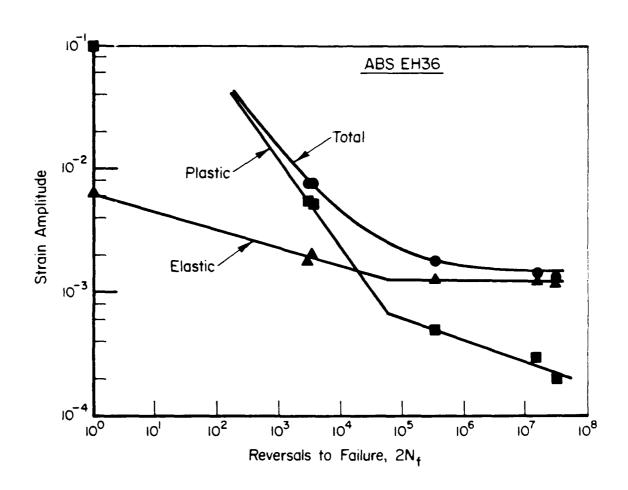


FIGURE 14. STRAIN-LIFE DATA FOR ABS EH36

## 7.3.3. Single Pore

The single pore geometry and assumed crack growth pattern are shown in Figure 15. The maximum pore size allowed for an isolated pore in the Rules for Nondestructive Inspection of Hull Welds  $^{\left[54\right]}$  is given as 0.25t or 0.1875 inch, whichever is less. The pore sizes chosen represent the largest allowable pore size and two larger sizes. The pore is assumed spherical and positioned at the centroid of the cross section. The crack growth pattern is assumed to remain circular throughout the crack propagation stage. The finite thickness correction factor,  $\mathrm{M}_{\mathrm{t}}$ , for a circular crack is approximated by the polynomial expression

$$M_t = 1.46 - 1.85(a/(t/2)) + 1.79(a/(t/2))^2$$
 (16)

This expression is the result of a regression of solutions of different crack depths found on pages 294-295 in Rooke and Cartwright  $^{[21]}$  for elliptical cracks in a semi-infinite medium. The stress intensity solutions are presented in Figure 16. Note that the initial stress intensity factor is quite high. As the crack becomes larger and grows out of the region of influence of the stress gradient, the stress intensity value decreases.

The results of the fatigue life predictions are presented in Tables 5 and 6 and plotted as S-N curves in Figures 17-20.

# 7.3.4. Uniform Porosity

The uniform porosity geometry and assumed crack growth pattern are shown in Figure 21. The porosity is assumed to be uniformly distributed throughout the weld. The Rules for Nondestructive Inspection of Welds [54] states that no more than 1.5 percent area porosity is allowed. It also states that pores smaller than 0.015 inch may be disregarded. The smallest pore size chosen is therefore 0.015 inch. Two other larger pores are also considered for both thicknesses. The analysis assumes that the maximum allowable area percent porosity is always present throughout the weld. This reduction in net cross sectional area has the

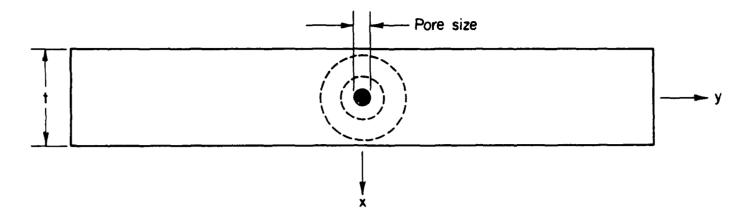


FIGURE 15. GEOMETRY AND ASSUMED CRACK GROWTH PATTERN (DASHED LINE) FOR SINGLE PORE

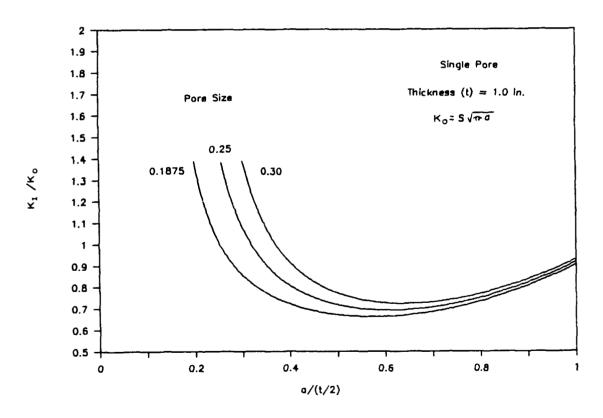


FIGURE 16. STRESS INTENSITY SOLUTION FOR SINGLE PORES IN A 1-INCH THICK PLATE

SINGLE PORE CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 0.5 INCH ABS EH36 TABLE 5.

Stress Ration-1 Residual Stress=51 ks. Str. 8 6 6	Stress Ratio=D Residual Stress=51 ksi Stre 4 30 31	Stress RationU.5 Residual Stress=51 ksi Stre	Stress Ration-1 Residual Stressou ksi Stre 6 6 4	Stress RationU Residual Stress-O ksi Stre Al	Stress Rationuls Residual StressHO Ksi Stre 20 10
Stress Range Casi) 81.60 61.20 40.80 20.40	Stress Range (ksi) 40.80 30.60 20.40 10.20	Stress Range (ksi) 20.40 15.30 10.20 5.10	Stress Range (ksi) 81.60 61.20 40.80 20.40	Stress Range (ksi) 40.80 30.60 20.40 10.20	Stress Range (ksi) 20.40 15.30 10.20 5.10
P N-Init 2590 8835 79753 32537876	N-Init 25725 245351 13532427	P-Init 3346237	P-Init 9766 48345 942751	P. Init 122443 1867610	P.Init 22816405
Pore=0.125 N-Prop 10709 39081 242320 5482800	Pore=0.125 N-Prop 121167 442150 2741500	Pore=0.125 N-Prop 1370860	Porce0.125 M-Prop. 10709 39081 242320	Pore=0.125 N-Frop 121167 442150	Pore 10, 125 N-Prop 1370860
inch N-TOTRL 13299 47916 322073 38020676	inch N-TOTAL 146892 687501 16273927	inch N-TOTAL 4717097 >100000000 >1000000000	inch H-TOTAL 20475 87426 1185071 >1000000000	inch N-F0TBL 243610 2309760 100000000 1000000000	inch N-TOTAL 24187265 >1000000000 >1000000000
N-Init 2362 7971 68868 25656872	M-Init 22024 201521 10576766	N-Init 2585648 78718952	N-Init 8691 42170 772719	N-Init 101635 1469932	N-Init 17001351
Pore=0.1875 N-Prop 3983 14538 90120 2039600	Pore=U.1875 N-Prop 45066 164480 1019800	Pore=0.1875 N-Prop 509860 1860800 >	Pore=0.1875 N-Prop 3983 14536 90120	Pore=1, 1875 N-Prop 45066 164480	Pore=0.1675 M-Prop 509860 >
inch N-TOTAL 6345 22509 158988 27696472	5 inch N-rorAL 67090 366001 11596566 >1000000000	5 inch N-TOTAL 3095508 80579752 >100000000	5 inch N-TOTHL 12674 56708 862839 >1000000000	5 inch N-TOTAL 146701 1634412 >100000000000000000000000000000000000	5 inch N-TOTRL 17511211 >100000000 >100000000
N-Init 2251 7554 63821 22652879	N-Init 20317 181869 9285158	P N-Init 2258772 67927084	N-Init 8175 39270 696953	N-Init 92270 1297597	P N-Init 14573639
Pore=0.250 N-Prop 1362 4968 30800 696900	Pore=0.250 N-Prop 15405 56200 348500	Pore=0.250 N-Prop 174280 635800	Pore=0.250 N-Prop 1362 4968 30800	Pore=0.250 H-Prop 15405 56200	Pore=0.250 N-Prep 174280
inch N-TOTAL 3613 12522 94621 23349779	inch N-TOTRL 35722 238069 9633658 >1000000000	inch N-TOTRL 2433052 68562894 >1000000000	inch N-TOTAL 9537 44238 727753	inch N-TOTAL 107675 1353797 >100000000	inch N-TOTAL 14747919 >100000000 >100000000

SINGLE PORE CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 1.0 INCH ABS EH36 TABLE 6.

inch N-TOTAL 5247 18481 130413 22815050	inch N-TOTAL 54009 298522 9481033 >1000000000	inch 2498161 2498161 64402335 >1000000000	inch N-TOTAL 10974 48998 730269	inch N-TOTAL 122344 1343513 • JUGOUGGGO	1 nch N-T0TAL 13858689 >1000000000 >1000000000
Pore=0.300 N-Prop 3051 11131 69020 1561690	Pore=0.300 N-Prop 34511 125938 780870	Pore=0,300 N-Prep 390450 1424820	Pore=0,300 N-Prop 3051 11131 69020	Porcett, 300 N-Prop 34511 125938	Pore=0,300 N-Prop 390450 ,
N-Init 2196 7350 61393 21253360	N-Init 19498 172584 8700163	N-Init 2107711 62977515	N-Init 7923 37867 661249	N-Init 87833 1217575	N-Init 13468239
inch N-TOTAL 6754 23983 165691 24958009	inch N-TOTRL 71253 367746 10437668	inch 2835042 70030134 >100000000	inch N-TOTRL 12678 55699 798823	inch N-TOTHL 143206 1483474 >1000000000	1mch N-TOTHL 15.149909 >1000000000 >1000000000
Pore=0.250 N-Prop 4503 16429 101870 2305130	Pore=0.250 N-Prop 50936 185877 1152510	Pore=0.250 N-Prop 576270 2103050	Porg=0.250 N-Prop 4503 16429 101870	Porc=0.250 N-Frop 50956 185877	Poresu.250 R-Prop S76270
M-Init 2251 7554 63821 22652879	N-Init 20317 181869 9285158	N-Init 2253772 67927084	N-Init 8175 39270 696953	N-Init 92270 1297597	N-Init 14573639
inch N-TOTAL 9685 34693 234545 29405972	5 inch 104865 503854 12451166	5 inch 3522878 82139472 >1000000000	5 inch N-TOTAL 16014 68892 938396 >100000000	5 1105h 184976 1772265 51000000000	5 11nch N=F0FHL 17938591 110000000000 11000000000000000000000
Pore=0.1875 N-Frop 7323 26722 165677 3749100	Pore=0.1875 N-Prop 92841 302333 1874400	Pore=0.1875 M-Prop 937230 3420520 >	Pore=0.1875 N-Prop 7323 26722 165677	PorvacO.1875 N-Frop 82841 302333	For e = 0, 1875 N-Prop 197230
N-Init 2362 7971 68368 25656872	N-Init 22024 201521 10576766	N-Init 2585648 78718952	N-Init 421.0 772719	M-Init 101635 1469932	N-Ioit 1700/1351
Stress Range (ksi) 81.60 61.20 40.80 20.40	Stress Range (ksi) 40.80 30.60 20.40 10.20	Stress Range (ksi) 20.40 15.30 10.20 5.10	Stress Range (ksi) 81.60 61.20 40.80 20.40	Stress Range (ksi) 40.80 30.60 20.40 10.20	Stress Range (1st) 20.40 15.30 10.20 5.10
Stress Ration-1 Residual Stress=51 ksi	Stress Ratio=O Residual Stress=51 ksi	Stress Ratio=0.5 Residual Stress=51 ksi	Stress Ration-1 Residual Stressof ksi	Stress RatiosA Residual Stress=0 ksi	Stress Ratio=0.5 Residual Stress=0 Est

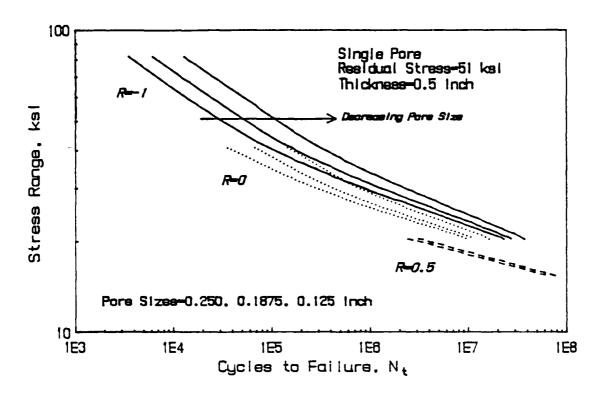


FIGURE 17. S-N CURVES FOR SINGLE PORE GEOMETRY IN 0.5-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

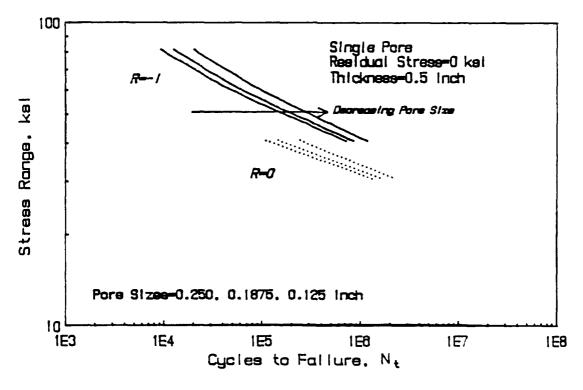


FIGURE 18. S-N CURVES FOR SINGLE PORE GEOMETRY IN 0.5-INCH THICK PLATE AND ZERO RESIDUAL STRESS

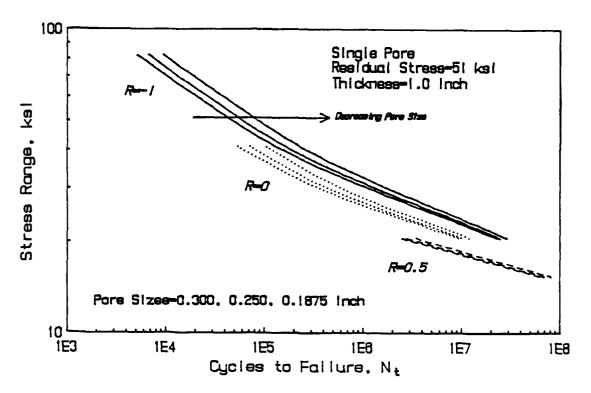


FIGURE 19. S-N CURVES FOR SINGLE PORE GEOMETRY IN 1.0-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

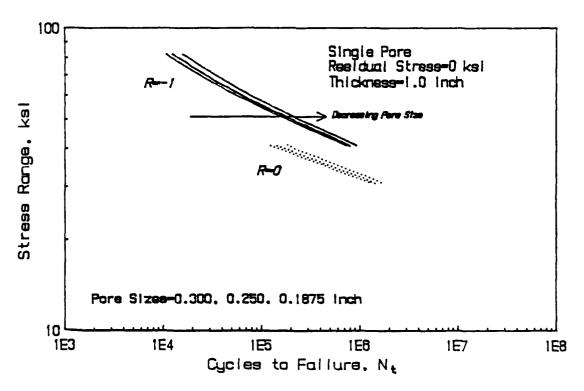


FIGURE 20. S-N CURVES FOR SINGLE PORE GEOMETRY IN 1.0-INCH THICK PLATE AND ZERO RESIDUAL STRESS

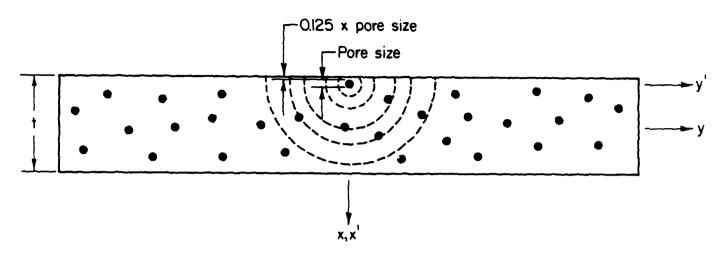


FIGURE 21. GEOMETRY AND ASSUMED CRACK GROWTH PATTERN (DASHED LINE) FOR UNIFORM POROSITY

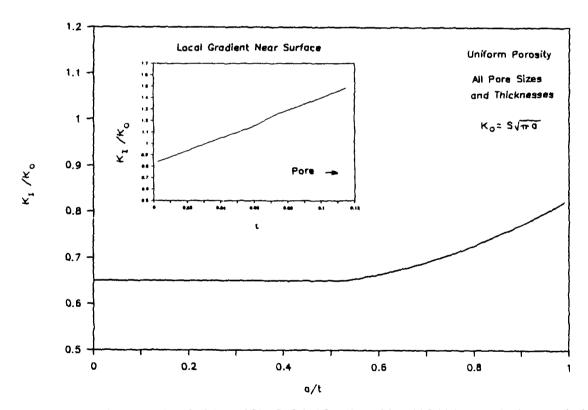


FIGURE 22. STRESS INTENSITY SOLUTION FOR UNIFORM POROSITY. INSET SHOWS THE DECAY OF THE STRESS INTENSITY AS THE CRACK GROWS AWAY FROM THE PORE STRESS GRADIENT TOWARD THE SURFACE

effect of raising the net section stress. (This assumption is not made for the other three geometries where the area reduction caused by the porosity is considered as negligible.)

The critical pore in this particular analysis is located in close proximity to the surface of the weldment. The elasticity result of Tsuchida and Nakahara [15] for a pore located 0.125 times the pore size (diameter) from the surface (a = 0.8 in Figure 4) is used to calculate the stress gradient to the surface. Since the pores relation to the surface causes an increase in the stress concentration, it is assumed that this pore will initiate a fatigue crack first. As this crack becomes the dominant singularity, no other cracks initiate. The stress intensity solution for the gradient near the surface is shown in the inset in Figure 22. The stress intensity steadily decreases until the crack breaks the surface. This near surface crack growth is assumed remain circular. When the crack intersects the near surface, the stress intensity solution is approximated as that of a semicircular crack in a slab. The stress intensity solution for this crack geometry is also found in [21] (page 298) and is represented by the expression

$$M_t = 0.70 - 0.34(a/t) + 0.47(a/t)^2$$
 (17)

where a is the crack radius and t is the plate thickness. The stress intensity solution for this geometry is shown in Figure 22.

The results of the fatigue life calculations are presented in Tables 7 and 8 and as S-N curves in Figures 23-26. Many of the cases which were analyzed proved to be non-propagating cracks, especially the small pores and high stress ratios.

# 7.3.5. Co-linear Porosity

The pore geometry and assumed crack growth pattern for the co-linear pores are shown in Figure 27. Lundin  $^{[17]}$  indicates linear or aligned porosity is usually associated with a root or interpass and found in concert with lack of penetration or fusion. Caution should therefore be exercised when trying to ascertain the structural integrity of a weldment

UNIFORM POROSITY CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 0.5 INCH ABS EH36 TABLE 7.

inch	inch	inch	inch	inch	inch
N-TOTAL	N-TOTAL	N-101RL	N-F07flL	N-TOTHL	R-TOTAL
91661	1062806	16205138	93690	1046677	17211366
333770	4930725	48485168	343438	3930913	86248644
>100000000	>100000000	>100000000	>1000000000	>1000000000	>100000000
Pore=0.045 i N-Prop 90644 330553	Pore=0.045 i N-Prop 1025328 3738520 >1	Pore=0.045 i N-Prop 15941800 42312600	Porezü, 045 y N-Prop 90644 330553	Pore=0.045	Pore=0.045 N-Prop 15941600 42312600
A-Init	M-Init	M-Init	M-Init	N-Init	N~Init
1017	37478	263338	3046	21349	1269566
3217	1192205	6172568	12885	192393	45936044
inch R-TOTBL 144927 529526 >1000000000	inch 1635532 6001242 >1000000000	inch N-TOTRL 19004588 83067923 >1000000000	inch N-TOTRL 146030 544898 >1000000000	1rich N-TOTRL 1663263 6331093 >100000000	100000000
Pore=0.030 : N-Prop 143530 525029	Pore=0.030 N-Prop 1626020 5932650	Pore≈0.030 N-Prαp 18395470 67131600 >	Pore≃0.030 N-Prop 143530 525029	Poreco.030 N-Prop 1626020 5932650	Fore=0.030 N-Prop 18595470 67131600
N-Init	M-Init	N-Init	N-Init	N-Init	N-Init
1397	9512	609118	4500	37243	3283635
4497	68592	15936323	19869	398443	136329370
inch	inch	inch	inch	inch	1100
N-TOTAL	N-FOTRL	N-TOTAL	N-TOTAL	N-TOTAL	N - 1 0 CHL
320921	36,18638	44596252	328715	3728686	E63 19966
>100000000	>100000000	>100000000	>100000000	-1000000000	> 1000000000
>100000000	>100000000	>100000000	>100000000	>1000000000	- 10000000000
Pore=0.015	Pore=0.015	Pore=0.015	Pore=0.015	Pore≃0.015	Pores0.015
N-Prop	N-Prop	N-Prop	N-Prop	N-Prop	N-Prop
318171	3590119	40632010	318171	3590119	40632010
N-Init	N-Init	N-Init	N-Init	N-Init	M-Im t
2750	28519	3964242	10544	138567	27607456
(ksi)	(ksi)	(ksi)	(ksi)	(ksi)	(ksi)
Stress Range (ksi)	Stress Range (ksi)	Stress Range (ksi)	Stress Range (ksi)	Stress Range (Ksi)	Stress Range (ksi <sup>1</sup> )
81.6	40.8	20.4	81.6	40.8	20.4
61.2	30.6	15.3	61.2	30.6	15.3
40.8	20.4	10.2	40.8	26.4	10.2
20.4	10.2	5.1	20.4	10.2	5.1
Stress Ratios-1	Stress Ratio=O	Stress Ratio=0.5	Stress Retion-1	Stress Rutio=0	Stress Ratio=0.5
Residual Stress-51 ksi	Residual Stress=51 ksi	Residual Stress≈51 ksi	Residual Stressou ksi	Residual Stress=0 ksi	Residual Stress-0 ksi

UNIFORM POROSITY CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 1.0 INCH ABS EH36 TABLE 8.

inch N-TOTAL 47654 173573 1075442	inch N-TOTAL 534728 1958021 12573669	inch 6124435 24426778 >100000000	1nch N-T0TRL 49004 179818 1146484	1nch N-TOTAL 543706 2036023 17193325 >100000000	1nch H-TOTHL E529448 38391900 >1000000000
Pore=0.075 N-Prop 46906 171234 1061765	Pore=0.075 8-Prop 530838 1936353 12007410	Pore=0.075 N-Prop 6003703 2191063U	Porc=0.075 N-Prop 46906 171234 1061765	Pore=0.075 N-Prop 530838 1936353 12007410	Pore=0.075 N-Prop 6003703 21910630
N-Init 748 2339 13677	N-Init 3890 21668 566259	N-Init 120732 2516148	N-Init 2098 8584 8584	N-Init 12868 99670 5185915	N-Init 525745 16481270
1 nch N-TOTAL 84869 308977 >100000000	1nch N-TOTAL 954457 3495941 >1000000000	1nch 10388597 45313828 >1000000000	1rich N-TOTRL 86898 318645 1000000000	1rich N-TOTAL 969846 3650856 1000000000	11994825 85077304 1000000000
Pore=0.045 N-Prop 83852 305760	Pore=0.045 M-Prop 948497 3458463	Porc=0.045 N-Prop 10725253 39141260	Pore=0.045 N-Prop 83852 305760	Pore=0.045 N-Prop 948497 3458463	Por c=0.045 N-Prop 10725259 39141260
N-Init 1017 3217	N-Init 5960 37478	M-Init 263338 6172568	N-Init 3046 12685	M-1814 21349 192393	M-1m1t 1269566 45936044
1msh N+T0TAL 304526 -100000000 -10000000000000000000000000	10000000000000000000000000000000000000	1000 A 2439472 42439472 100000000 1000000000 7 1000000000	inch N-TOTAL 312320 1000000000 11000000000 -1000000000	1000 N-FOTAL 1000000000 1000000000 100000000000000	10ch 662,3196 1000,00000 1000,00000 1000,00000
Pore=8.015 N-Prop 301776	Pore=0.015 N-Prop 3404936	Por∉=0.015 N-Prop 38535230	Por∉=0.015 N-Frop 30177€	Porv=0.015 N-Prop 3404936	Por w=0.015 N Prop 3.6535230
N-Intt 2750	N-Init 28519	N-Init 3964242	N-I m t 10544	H-Inat 138567	H Int 4
Stress Range (ksi) 81.6 51.2 40.8 20.4	Stress Pange (ks) 40.8 30.6 20.4 10.2	Stress Range (ks)) 20.4 15.3 10.2 5.1	Stress Range (ksi) 81.6 61.2 40.8	Stress Range Usas 40.8 20.8 20.4 10.2	Stress Bange (ks): 20.4 15.3 10.2 5.1
Stress Ration-1 Residual Stress=51 ksi	Stress Ratio=O Residual Stress=51 ksi	Stress Ratios0.5 Residual Stress:51 ks:	Stress Mation-1 Residual Stress=0 ksi	Stress RatiosO Residual Stressoultsi	Stress Ratio-0.5 Residuel Stressollisi

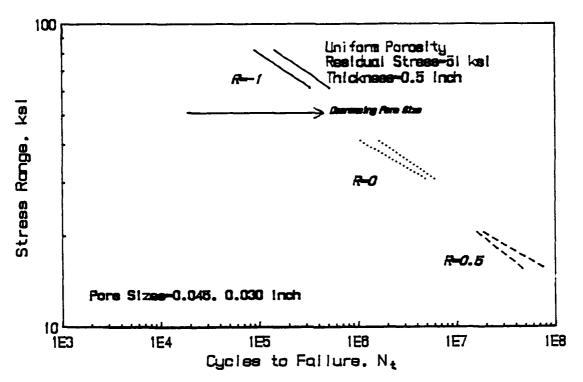


FIGURE 23. S-N CURVES FOR UNIFORM POROSITY GEOMETRY IN A 0.5-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

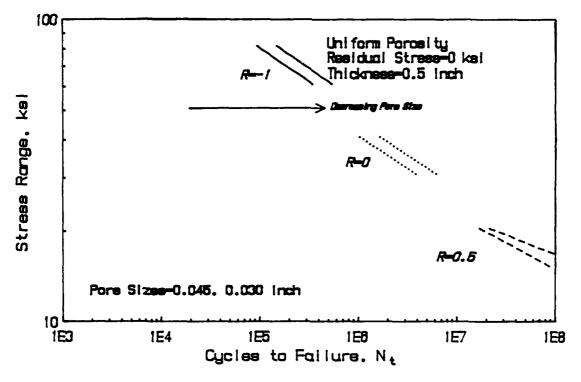


FIGURE 24. S-N CURVES FOR UNIFORM POROSITY GEOMETRY IN A 0.5-INCH THICK PLATE AND ZERO RESIDUAL STRESS

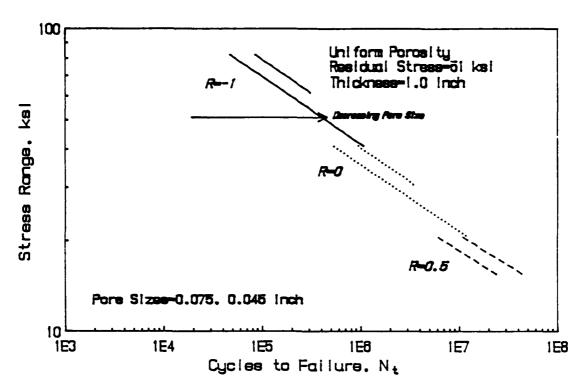


FIGURE 25. S-N CURVES FOR UNIFORM POROSITY GEOMETRY IN A 1.0-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

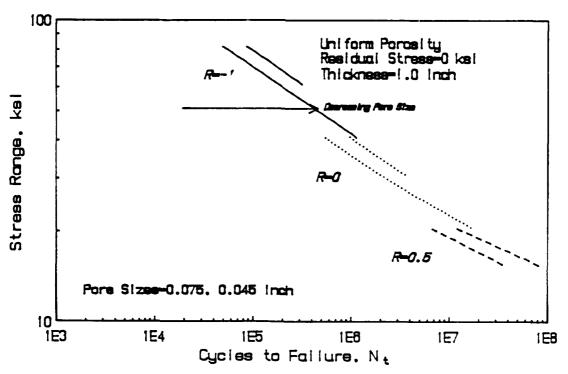


FIGURE 26. S-N CURVES FOR UNIFORM POROSITY GEOMETRY IN A 1.0-INCH THICK PLATE AND ZERO RESIDUAL STRESS

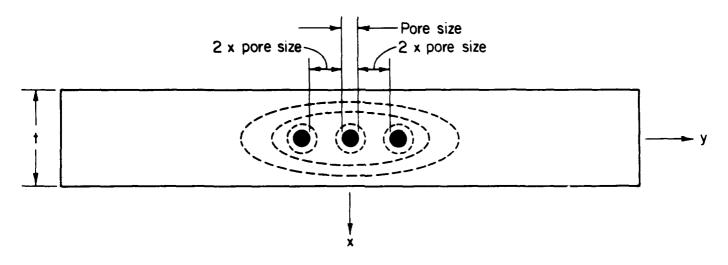


FIGURE 27. GEOMETRY AND ASSUMED CRACK GROWTH PATTERN (DASHED LINE) FOR CO-LINEAR PORES

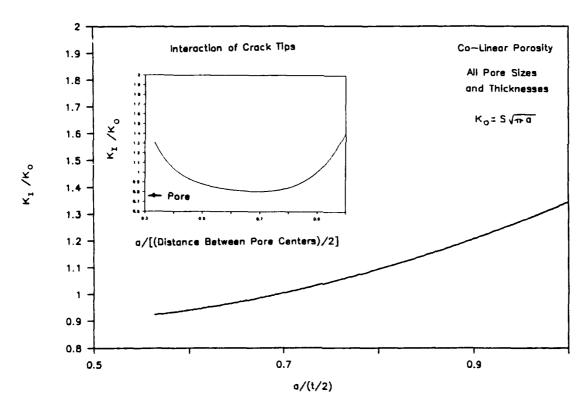


FIGURE 28. STRESS INTENSITY SOLUTION FOR CO-LINEAR POROSITY. INSET SHOWS THE RISE IN STRESS INTENSITY AS THE CRACK TIPS FROM INDIVIDUAL PORES APPROACH EACH OTHER

containing co-linear porosity based upon the pores alone. Assuming that the weld may have a significant crack initiation period may be highly unconservative if a planar defect such as lack of penetration is present. The analysis technique presented here does not account for any planar defects and should be considered in the light of the foregoing comments.

The pores are initially spaced two pore diameters apart so no stress gradient interaction is assumed. The cracks initiating from the pores are assumed to occur at nearly the same time and grow simultaneously. Before the individual circular cracks join, there will be interaction between the approaching crack tips resulting in an increased stress intensity factor and accelerated crack growth. No stress intensity solution was available for two co-planar cracks in a three dimensional medium so this interaction was approximated by the solution two dimensional sheet solution [21]. The solution is represented by the polynomial expression

$$M_{co} = 1 + 0.88(a/d) - 6.6(a/d)^2 + 23.3(a/d)^3 - 32.9(a/d)^4 + 16.6(a/d)^5$$
 (18)

where a is the crack radius and d is the distance between pore centers. The stress intensity solution is shown in the inset in Figure 28. This assumption is conservative although somewhat tempered by the crack shape factor  $\Phi_0$  in Equation (13). For a circular crack,  $\Phi_0$  is 1.57 which reduces the stress intensity by about 0.6.

After the individual circular cracks join, the crack shape becomes elliptical (a/c equals approximately 0.4) and growth continues. As with the circular cracks, the elliptical crack is assumed to undergo self-similar growth. This assumption is less accurate since elliptical cracks actually tend to grow into the more energetically stable circular shape. The  $\rm M_t$  correction factor for the elliptical crack is again found in  $^{[21]}$  (pages 294-295) and is approximated by

$$M_{t} = 1.22 - 1.10(a/(t/2)) + 1.40(a/(t/2))^{2}$$
 (19)

The stress intensity solution is plotted in Figure 28. The results of the fatigue predictions are given in Tables 9 and 10 and as S-N curves in Figures 29-32.

### 7.3.6. Cluster Porosity

The pore geometry and assumed crack growth pattern for the cluster porosity analysis is shown in Figure 33. The cluster porosity is the most difficult to model analytically because of the infinite variety of pore sizes and configurations which clusters can assume. This variety is apparent from the fracture surface photographs in Figure 8. The geometry for the analysis presented here was chosen to model the three dimensional nature of clusters (not all pores on the same plane) and the possibility of interaction between individual clusters. The individual pores are all equal size and are assumed to initiate a crack at the same time. They are spaced a distance of 0.25 times the individual pore size so the stress gradients will interact (see Figure 5). The interaction results in an increased stress concentration factor and, therefore, fatigue notch factor.

The initiation life for the clusters consists of two stages: individual pore cracking coalescence; and initiation of a crack around the periphery of the cluster. Because the stress concentration factor is higher for the material toward the center of the cluster due to interaction, that material is more severely damaged compared to the material on the periphery of the cluster. The cycles to coalescence is calculated using the higher, interaction-influenced, fatigue notch factor. Meanwhile the periphery material has accumulated a lesser amount of fatigue damage although not enough to have initiated cracking. Using the Palmgren-Miner linear damage rule,

$$\sum \frac{N(\text{at stress level x})}{N(\text{failure at stress level x})} = 1 \text{ at failure}$$
 (20)

CO-LINEAR POROSITY CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 0.5 INCH NUMBER OF PORES = 3 ABS EH36 TABLE 9.

inch N-TOTAL 3899 13570 101121 23496979	inch N-FOTAL 38967 24939 9707158	inch N-TOTRL 2469772 68697184 >1000000000	inch N-TOTAL 9823 45286 734253 >100000000	inch R-TOTAL 110920 1365667 >100 300000	inch N-TOTAL 14784639 -1000000000 >10000000000000000000000000
Pore=0.250 N-Prop 1648 6016 37300 844100	Pore=0.250 N-Prop 18650 68070 422000	Pore=0.250 M-Prop 211000 770100	Pore=0.250 N-Prop 1648 6016 37300	Pore=0.250 N-Prop 18650 68070	Pore=0.250 N-Prop 211000
N-Init 2251 7554 7554 63621 22652879	M-Init 20317 191869 9285158	M-Init 2258772 67927084	N-Init 8175 39270 696953	N-Init 92270 1297597	N-Init 14573639
inch N-TOTHL 5669 20041 143688 27350272	5 inch N-TOTAL 53440 338081 11423266 >1000000000	5 inch M-TUTAL 3003008 80263952 >1000000000	5 inch N-TOTAL 11998 54240 847539	5 inch N-TUTBL 139051 1606492 >100000000	5 inch N-TUTAL 17424661 >1000000000 >1000000000
Pore=0.1875 N-Prop 3307 12070 74820 1693400	Pore=0.1875 N-Prop 37416 136560 846500	Pore=0.1875 N-Prop 423310 1545000	Pore:0.1875 N-Prop 3307 12070 74820	Poreso, 1875 M-Prop 37416 136560	Pore=0, 1375 M-Prop 423310
N-Init 2562 7971 68868 25656872	N-Init 22024 201521 10576766	N-Init 2585698 78718952	N-Init 8691 42170 772719	N-Init 101635 1469932	M-Init 17001351
inch N-TOTAL 8442 30194 212189 35534206	inch N-TOTAL 91924 486996 15030787	inch N-TOTAL 4095187 >100000000 >1000000000	inch M-rorAL 15618 69704 1075187	inch H-FDTAL 188642 2109255 >1000000000	inch N-TOTAL 23565355 >1000u00000 >1000u00000
Porw=0.125 N-Frop 5852 21359 132436	Pore=0.125 N-Prop 66139 241645 1498360	Pore=0.125 N-Prop 748950 >	PorceO.125 N-Prop 5852 21359 132436	Porect. 125 N-Prop 66199 241645	Pore=0.125 N-Prop 748950
N-Init 2590 8835 79753 32537876	N-Init 25725 245351 13532427	N-Init 3346237	M-Init 9766 48345 942751	M-Init 122443 1867610	N-Init 22816405
(ksi)	(ksi)	(ksi)	(ksi)	(ks1)	G.
Stress Range (ksi) 81.60 61.20 40.80 20.40	Stress Range (ksi) 40.80 30.60 20.40 10.20	Stress Range (ksi) 20.40 15.30 10.20 5.10	Stress Range (4si) 81.80 61.20 40.80 20.40	Stress Range (kst) 40.80 30.60 20.40 10.20	Stress Range (181) 20.40 15.30 10.20 5.10
. Ks	- k31	Ksi	51	ភ ភ	K51
Stress Ratio=-1 Residual Stress≃51 ksi	Stress Ratio=O Residual Stress=51 ksi	Stress Ratio=U.5 Residual Stress=51	Stress Ratio=-1 Residual Stress=O ksi	Stress RatiosO Residual Stress=O ksi	Stress Ratio=0.5 Residual Stress=0
ഗ് മ്	দ ঐ	ଉଚ୍ଚି	ശ്യ്	ന്ത്	ଫରି

CO-LINEAR POROSITY CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 1.0 INCH NUMBER OF PORES = 3 ABS EH36 TABLE 10.

inch N-TOTAL 4099 14293 104441 22227350	inch N-TUTRL 41022 251129 9187193 >1000000000	inch N-TOTAL 2351235 63866145 >1000000000	inch N-TOTRL 9826 44810 704297 >1000000000	inch M-TOTRL 109357 1296120 >1000000000	1nch N-T0TAL 13711763 >1000000000 >1000000000
Pore=0.300 N-Prop 1903 6943 43048 973990	Pore=0.300 N-Frop 21524 78545 487030	Pore=0.300 N-Prop 243524 888630 ,	Pore=0.300 N-Prop 1903 6943 43048	Pore=0.300 N-Prop 21524 78545	Pore=0.300 N-Prop 243524
N-Init 2196 7350 61393 21253360	M-Init 19498 172584 8700163	N-Init 2107711 62977515	M-Init 7923 37867 661249	N-Init 87833 1217575	H-Init 13468239
inch N-TOTRL 4713 16532 119486 23912329	inch N-TOTBL 48151 283435 9914937 >1000000000	inch 2573685 69076164 >100000000	inch N-TOTAL 10637 48248 752618 >100000000	inch N-TOTAL 120104 1399163 >100000000	inch H-TOTAL 1488552 >100000000 >100000000
Port = 0.250 N-Frop 2462 8978 55665 1259450	Pore=0.250 N-Prop 27834 101566 629779	Pore=0.250 N-Prop 314913 1149080	Porez0.250 N-Prop 2462 8978 55665	Port=0.250 N-Frop 27834 101565	Pore=0.250 N-Prop 314913
N-Init 2251 7554 63821 22652879	M-Init 20317 181869 9285158	M-Init 2258772 67927084	M-Init 8175 39270 696953	N~Init 92270 1297597	N-Init 14573639
inch M-TOTAL 6146 21277 154463 27594022	5 inch N-TOTAL 64825 357736 11545162	5 inch N-TOTAL 3069867 80486332 >1000000000	5 inch N-TOTAL 12475 55976 858314	1 inch N-TOTAL 144436 1626147 1000000000 10000000000	11/35h N-TOTAL 17435570 100000000 100000000
Pore=0.1875 N-Prop 3784 13806 85595 1937150	Porce 0.1875 N-Prop 42801 156215 968396	Pore=0.1875 N-Prop 484219 1767380	Pore=0.1875 N-Prop 3784 13806 85595	Porg=0.1875 N-Prop 42801 156215	Por e 20, 1875 N-Prop 484219
N-Init 2362 7971 68868 25656872	M-Init 22024 201521 10576766	N-Init 2585648 78718952	M-Init 8691 42170 772719	N-1 na t 1010-35 1469432	8-Int 17001 (51
(k <b>s1</b> )	(ksi)	(ksi)	( <del>s)</del>	(ks1)	d sto
Stress Range (ksi) 81.6 61.2 40.8 20.4	Stress Range (ksi) 40.8 30.6 20.4 10.2	Stress Range Cksi) 20.4 15.3 10.2 5.1	Stress Range (ksi) 81.6 61.2 40.8 20.4	Stress Range 40.8 30.6 20.4 20.4 10.2	Stress Range if sto 2014 1513 1012 811
Stress Ration-1 Residuel Stress=51 ksi	Stress RatiosB Residual Stress=51 ksi	Stress Rational.5 Residual Stress=51 ksi	Stress Rational Residual Streuscollesi	Stress RationA Residual Stress O kei	Stress Rotherdis Ferndual Stress-U.Es.

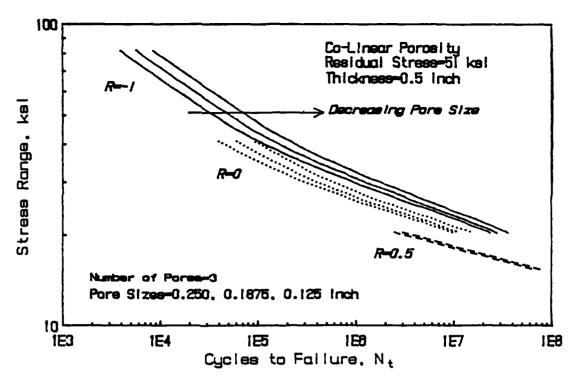


FIGURE 29. S-N CURVES FOR CO-LINEAR POROSITY GEOMETRY IN A 0.5-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

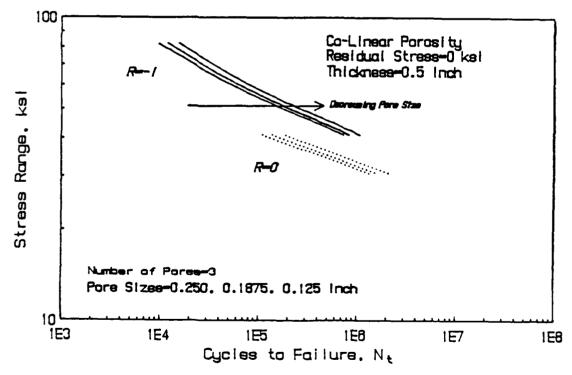


FIGURE 30. S-N CURVES FOR CO-LINEAR POROSITY GEOMETRY IN A 0.5-INCH THICK PLATE AND ZERO RESIDUAL STRESS

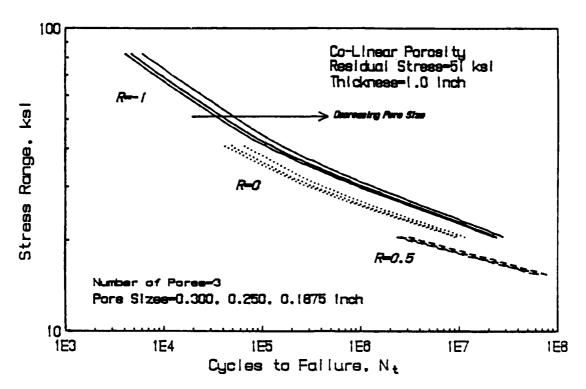


FIGURE 31. S-N CURVES FOR CO-LINEAR POROSITY GEOMETRY IN A 1.0-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

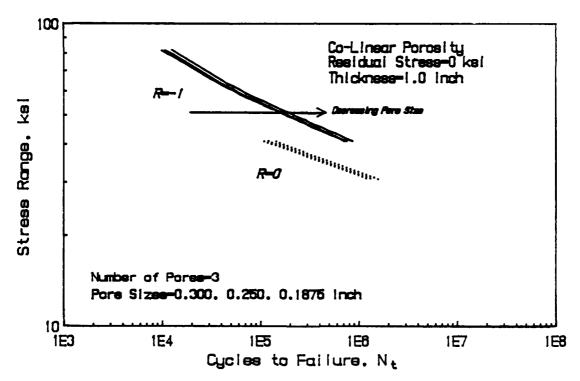


FIGURE 32. S-N CURVES FOR CO-LINEAR POROSITY GEOMETRY IN A 1.0-INCH THICK PLATE AND ZERO RESIDUAL STRESS

where N denotes cycles, the outer material has been damaged an amount

N(coalescence)
N(failure a periphery stress level)

Before initiating a fatigue crack, the outer material must satisfy Miner's criteria (Equation 20). After the inner region of the pores coalesce, the load path around the cluster will change because load can no longer be carried between the pore ligaments. Although the stress field around the cluster will admittedly be very complex, it is assumed for our purposes to approximate the stress field around an ellipsoid of comparable dimensions. Observing Figure 33, the ellipsoid will be an oblate spheroid. half as high as it is wide. In reference to Figure 3, it would be of the shape a=b=1 and c=0.5. The remaining initiation life of the cluster (before a crack begins growing radially) at this new higher stress concentration level is calculated from Equation 20. The total initiation life is taken as the cycles to cause coalescence and the cycles remaining before the periphery initiates a crack. The crack growth stress intensity solution is shown in Figure 34. Note the high initial stress intensity factor. This is due to the high stresses resulting from the assumed ellipsoid shape of the coalesced cavity. The stress intensity factor decays rapidly and the solution becomes dominated by the  $\mathbf{M}_{t}$  factor. This is the same as the single pore  $\mathbf{M}_{t}$  solution, Equation 16, because both are circular cracks.

The fatigue life predictions for the cluster geometry are presented in Tables 11 and 12 and as S-N curves in Figures 35-38.

## 8. VARIABLE AMPLITUDE LOADING

# 8.1. SL-7 Containership Instrumentation Program

The SL-7 instrumentation program performed by the U.S. Navy and the U.S. Coast Guard produced a vast amount of stress history data on ocean going vessels. The seven year program (1972-1980) collected midship bending stress data from eight SL-7 high speed container ships on both

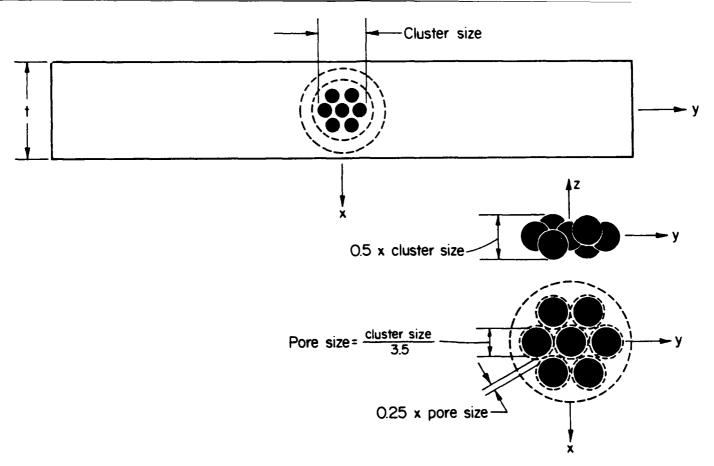


FIGURE 33. GEOMETRY AND ASSUMED CRACK GROWTH PATTERN (DASHED LINE) FOR CLUSTER POROSITY

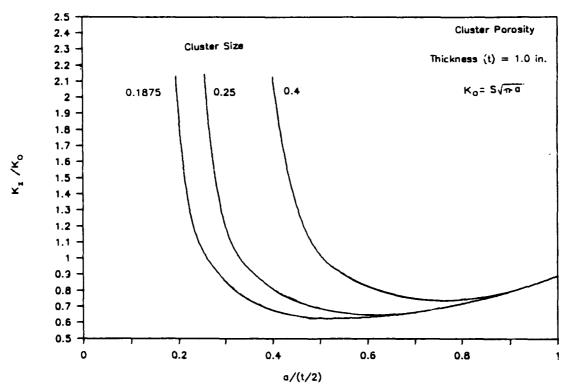


FIGURE 34. STRESS INTENSITY SOLUTION FOR CLUSTER POROSITY IN A 1.0-INCH THICK PLATE

CLUSTER POROSITY CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 0.5 INCH ABS EH36 TABLE 11.

inch N-TOTAL 2032 6707 57743 14026292	inch N-TOTAL 16036 137731 5649570 >1000000000	inch N-TOTAL 136.1844 37376792 >1000000000	inch R-TOTAL 6843 31458 487404 >1000000000	inch N-TOTAL 65892 815650 85668573	inch N-TOTAL 9130354 >100000000 >100000000
Pore=0.300 N-Prop 39 140 7379 171500	Pore=0.300 N-Prop 12936 85760	Fore=0.300 N-Prop 1315703 37222982	Pore=0.300 N-Frop 39 140 7379	Pore=0.300 N-Prop 133 12936 85760	Pore=0.300 N-Prop 1315703
N-Init 1993 6567 50364 13854792	N-Init 15603 124795 5563810	N-Init 46141 153810	N-Init 6804 31318 480025	N-Init 65459 802714 85582813	N-Init 7814651
inch N-TOTAL 2562 10568 133588 24066308	5 inch N-TOTAL 53393 313896 10005286	5 inch 2609349 68299781 >100000000	5 inch N-TOTAL 8810 43811 780412 >104 30000	5 inch R-TOTAL 127139 1431697 >1000000000	5 inch H-TOTAL 14728023 >100000000 >100000000
Pore=0.1875 N-Prop 146 2455 66133 1420820	Pore=0.1875 N-Prop 32036 128011 739750	Pore=0.1875 N-Prop 365790 1318920	Pore=0.1875 N-Frop 146 2455 66133	Pore=0.1875 N-Prop 32096 128011	Pore=0.1875 N-Prop 365790
M-Init 2416 2416 8113 67455 22645488	N-Init 21297 185885 9265536	N-Init 2244159 66980861	N-Init 8664 41356 714279	M-Init 95043 1303686	N-Init 14362233
1nch N-TOTRL 3265 36742 279341 43756443	inch N-TOTAL 126723 600819 18535956	inch N-TOTAL 5157340 >100000000 >1000000000	inch N-TOTAL 11662 83391 1320966	inch N-TOTAL 242965 2590749 100000000	1005h H-10THL 29720794 >100000000 100000000
Fores0.125 N-Frop 26403 184396 4206900	Pore=0.125 N-Prop 96121 306410 2011560	Fores0.125 N-Prop 1045220	Pore≃0.125 N-Frop 26403 184396	Porez0.125 M.Prop 96121 306410	Porest. 125 N-Frap 10452.0
M-Init 3001 10339 94945 39549543	N-Init 30602 294409 16524396	N-Init 4112120	N-Init 11398 56988 1136570	N-Init 146864 2284339	H - Ina t 288-75574
Stress Range (ksi) 81.60 61.20 40.80 20.40	Stress Range (ks) 40.80 30.60 20.40 10.20	Stress Range Cks1) 20.40 15.30 10.20 5.10	Stress Range (532) 81.60 61.20 46.80 20.40	Stress Range (4 st.) 40.80 30.60 20.40 10.50	Streets Kange (1930) 20, 40 15, 50 10, 20 5, 10
Stress Ration-1 Residuel Stress:51 ksi	Stress RatiosD Residual Stress=51 ksi	Stress RationO.5 Rasidual Stress=51 ksi	Stress Ration-1 Residual Stressoù Esi	Stress Batho H Residual Stress U Fai	Stress Batto U.S Residual Strossiú ksi

CLUSTER PORE CONSTANT AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 1.0 INCH ABS EH36 TABLE 12.

inch N-TOTAL 1878 6154 67259 11599031	inch N-TOTAL 19186 147899 465631	inch N-TOTRL 116624 28805388 >100000000	inch N-TOTRL 6139 27767 424091	inch N-TOTRL 60910 687355 64112743	inch R-TOTRL 6038945 >100000000 >1000000000
Pore=0.400 N-Prop 55 196 23137 557720	Pore=0.400 N-Prop 5640 43862 261050	Porc=0.400 N-Prop 138664 495850 >	Pore=0.400 N-Prop 55 196 23137	PorceO.400 N-Prop 5640 43862 261050	Porce 0.400 R-Prop 138664 >
M-Init 1823 5958 44122 11041311	N-Init 13546 104037 4395581	N-Init 1027960 28309538	N-Init 6084 27571 400954	N-Init 55270 643493 63851693	A-Init 5900281
inch H-TOTAL 2244 7472 138629 18335658	inch N-TOTAL 56204 295306 7575249 >1000000000	inch N-TOTAL 2027733 47476378 >100000000	inch N-TOTAL 7516 34905 633777	inch N-TOTRL 113314 1102012 >100000000	inch N-TOTRL 10108199 >1000000000 >10000000000
Pore=0.250 N-Prop 112 404 82902 1885270	Pore=0,250 N-Prop 38824 151976 925680	Pore=0.250 H-Prop 442058 1734370	Pore=0.250 : N-Prop 112 404 82902	Pore=0.250 : N-Prop 38824 151976 > >	Pore=0,250 3 N-Prop 442058
N-Init 2132 7068 55727 16450389	N-Init 17380 143330 5649569	M-Init 1585675 45742008	N-Init 7404 34501 550875	N-Init 74490 950036	N-Init 9866141
inch N-TOTAL 2567 12875 198268 25432808	5 inch N-TOTAL 85655 436996 10706906	5 inch N-TOTAL 2919479 69772191 >100000000	5 inch N-TOTAL 8815 46119 845092	5 inch N-TOTAL 159411 1554797 FIDGODODO	5 inch N=T0THL 29350894 1000000000 11000000000
PorezO.1875 N-Frop 151 4762 130813 2787320	Pore=0.1875 N-Prop 64368 251111 1441270	Pore=0.1875 N-Prop 675320 2791330 >	Pore=0.1875 N-Prop 151 4762 130813	Pore=0.1875 N-Prop 64368 251111	Porezu, 1875 N-Prop 875320 1
M-Init 2416 2416 8113 67455 22645488	M-Init 21297 185885 9265536	N-Init 2244159 66980861	N-Init 8664 41356 714279	N-Init 95043 1303686	M-Inat 28675574
Stress Range (ks1) 81.60 61.20 40.80 20.40	Stress Range dess) 40.80 30.60 20.40 10.20	Stress Range Cksi) 20.40 15.30 10.20 5.10	Stress Range (ksi) 81.60 61.20 40.80 20.40	Stress Range (Lsi) 40.80 30.60 20.40 10.20	Stress Range (4st) 20.40 15.30 10.20 5.10
Stress Ration-1 Residual Stress-51 ksi	Stress RatiosU Residual Stress=51 ks.	Stress RationN.5 Residual Stress=51 ksi	Stress Ratios-1 Residual Stressoo ksi	Stress RatiosN Residual Stress=0 ksi	Stress RatiosD.5 Residud Stress=O Vsi

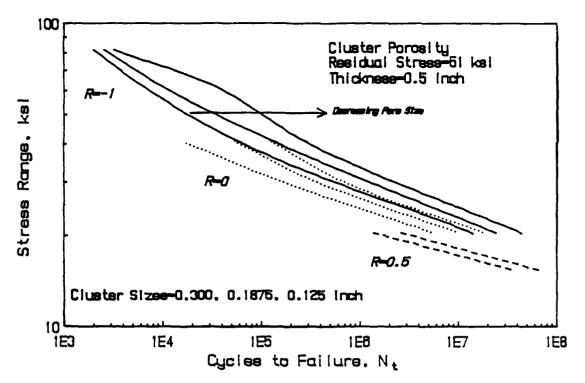


FIGURE 35. S-N CURVES FOR CLUSTER POROSITY IN A 0.5-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

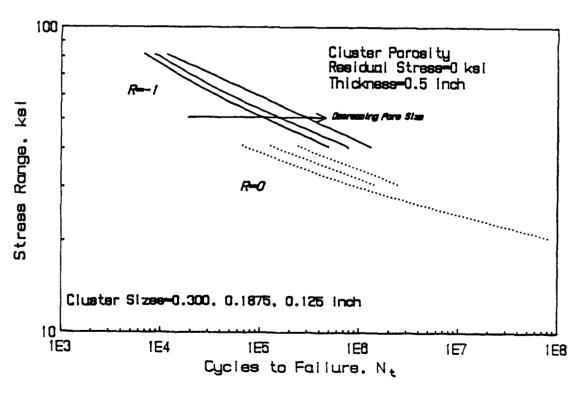


FIGURE 36. S-N CURVES FOR CLUSTER POROSITY IN A 0.5-INCH THICK PLATE AND ZERO RESIDUAL STRESS

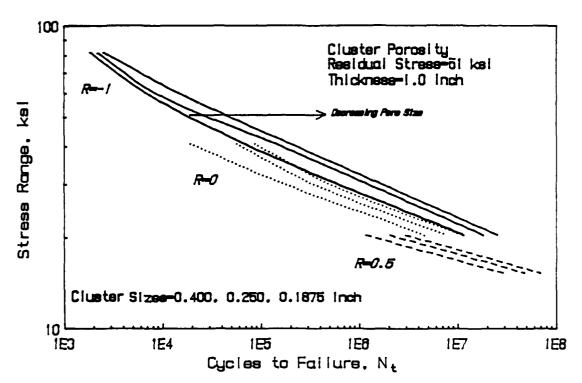


FIGURE 37. S-N CURVES FOR CLUSTER POROSITY IN A 1.0-INCH THICK PLATE AND 51 KSI RESIDUAL STRESS

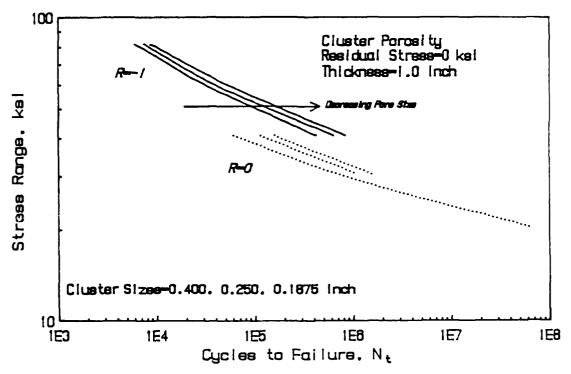


FIGURE 38. S-N CURVES FOR CLUSTER POROSITY IN A 1.0-INCH THICK PLATE AND ZERO RESIDUAL STRESS

transatlantic and transpacific routes. A sample of this data was used to generate a stress history to be used in the predictive model.

## 8.1.1. Data Characteristics

Stresses induced in a ship structural element have components from a number of sources. These include<sup>[12]</sup> local residual stress from fabrication or welding, initial still water bending stress, varying mean stress due to fuel burn off, the ships own wave system, diurnal thermal stresses, low frequency wave-induced stress, and high frequency wave induced stress. Of these only the wave induced stresses, both low and high frequency will be used in constructing a stress history for the model. The other sources will be considered as quasi-static, contributing to the instantaneous mean stress rather being than a source of cyclic loading.

High frequency wave induced stresses are caused by dynamic wave loading against the ship structure. These can consist of bottom slamming, shipping of water on deck, and flare impact. Dynamic loads produce whipping and springing elastic motions of the hull, typically at higher than the frequency of wave encounter. Low frequency wave-induced stresses occur at the same frequency as wave encounter. These are caused by the wave forces on the hull. The level of stress is directly related (although not directly proportional to) the significant wave height of the encountered seaway.

The stresses recorded during the SL-7 instrumentation program are the maximum peak stress and the maximum trough stress which occur during a four hour recording interval. These maximum stresses do not necessarily occur during the same cycle. In general, the maximum peak and trough stress recorded will be produced by a dynamic, high frequency load. Therefore, the majority of the reported data is high frequency data. A limited amount of low frequency data, however, has been reported  $\begin{bmatrix} 12 \end{bmatrix}$ . A representative history can be constructed from the available low and high frequency data.

The low frequency are directly related to the significant wave height encountered by the ship. The significant wave height is the

average height of the highest one third portion of the waves. Figure 39 illustrates the relation between the observed wave height and the root mean square (RMS) stress value. This data was collected on board the SL-7 SEA-LAND McLEAN during 1974; the first date year of the data collection program. The frequency of occurrence for each wave height is reported in [52] and presented in Table 13. From the loading summary sheets presented in Reference 12, the average number of wave cycles during a 20-minute interval is 176 cycles, or 385,440 cycles per month at sea. Using the cycle rate and the reported probability of occurrence for each wave group, a low frequency loading spectrum can be calculated based on RMS stresses.

The histogram<sup>[53]</sup> of maximum peak to trough stress recorded during date year one aboard the SL-7 SEA-LAND McLEAN (port) is shown Figure 40. Recall that each reported cycle is the maximum value, peak and trough, recorded during a 4-hour interval. The average rate of occurrence for high frequency or burst data is reported in Reference 12 as 18 bursts per 20-minute interval. This converts to 216 bursts for every one burst recorded. In constructing the high frequency portion of the loading spectrum, the conservative assumption will be made that 216 bursts occurred at the same value as the reported maximum. The number of cycles from the high and low frequency loadings are then combined on a per month basis as shown in Table 14. Any overlap of the high and low frequencies were assumed to be additive, i.e., an element of material will be damaged equally by a dynamic load and a low frequency load of equal magnitude.

# 8.2. Fatigue Predictions

Fatigue predictions were made using the same material properties and pore geometries as in the constant amplitude program. Reference 12 reported an average mean stress of 6.5 ksi. In service, the mean stress actually varies as fuel is spent and from ballast changes. Predictions were made at mean stress biases of 6.5 and 0. The stress history was scaled from 1 to 1.75 to provide a wide range of predicted service lives.

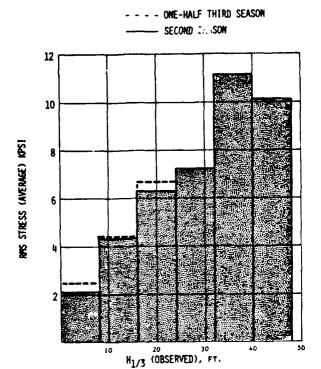


FIGURE 39. AVERAGE RMS STRESS VS. OBSERVED WAVE HEIGHT (AMIDSHIP BENDING STRESS). DASHED LINE REPRESENTS DATA FROM ONE-HALF OF THE THIRD SEASON. SOLID LINE REPRESENTS THE SECOND SEASON

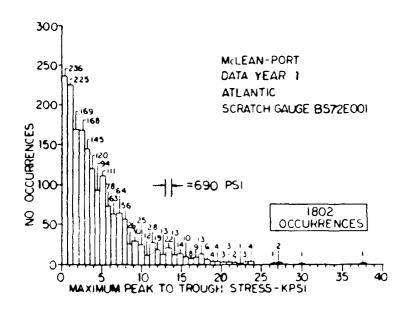


FIGURE 40. HISTOGRAM OF MAXIMUM PEAK TO THROUGH STRESS DURING DATA YEAR 1
ABOARD SL-7 MCLEAN (PORT)

TABLE 13. AVERAGE RMS STRESS BASED ON PROBABILITY OF OCCURRENCE FOR EACH WAVE GROUP

Wave Group	Probability of Occurrence of Wave Group	Average RMS Stress ksi
I	0.6294	2.037
II	0.3133	4.320
III	0.039	6.325
IV	0.0167	7.249
٧	0.0012	11.093
VI	0.0004	10.694

TABLE 14. VARIABLE AMPLITUDE LOADING SL-7 McLEAN YEAR ONE DATA ATLANTIC ROUTE

Stress Range (ksi)	Cycles/Month	Relative Frequency
2	261604	0.626
4.3	120758	0.289
6	23024	0.055
7.2	6437	0.015
10.2	3208	0.007
14	1296	0.003
18	864	0.002
22	432	0.001

The results are reported as blocks with each block representing 1 month of service at sea.

No attempt was made to employ a crack growth retardation model because the reported stress data consisted of either maximums recorded over a long time period (high frequency) or an averaged stress (low frequency). As such, no effect of the loading sequence can be accounted for.

#### 8.2.1. Results

The results of the variable amplitude fatigue life predictions are presented in Tables 15-22 and Figures 41-46. In general, the results for the history without being scaled (scale = 1) represent lives many times longer than any design lives, some on the order of thousands of years. For the uniform porosity case where the smallest pores were considered, some cracks were predicted to arrest after growing outside of the pore stress field. As the scale was increased, lives on the order of tens or hundreds of years were predicted.

## 9. PARAMETRIC DISCUSSION

The model used to predict the fatigue life of weldments containing porosity has been formulated to account for parameters which have been demonstrated to affect fatigue life. Some aspects of the model have been included based upon findings in the literature search dealing specifically with porosity, such as the need for pore interaction in pore clusters. The majority of the model's features are based upon historical precedent of linear elastic fracture mechanics and life predictions in notched specimens. In this section, the model's dependence upon the various parameters is examined. Referring to Table 3, the following parameters were varied in this study: thickness, residual stress, stress ratio, pore size, and porosity type. The features of the model which are influenced by these parameters will be highlighted with examples.

TABLE 15. SINGLE PORE
VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS
THICKNESS = 0.5 INCH
ABS EH36

inch N-TOTAL	76	237	1109	9643		ואנום				2158	
Pore=0.250 i t M-Prop	15	31	71	196	6 6	oren con	N-Frop	37	76	183	520
N-In	w	22	103	94.	í	<b>.</b>				1975	
inch N-TOTAL	116	334	1450	12106	•	ייטעיי	N~TOTAL	257	720	2904	22367
Pore≃O.1875 i N-Prop	. <del>4</del>	92	210	580		orest. Idro	N-Prop	111	237	555	1609
N-Inst	70	242	1240	11526	(	Ţ				2349	
nnch M-TüTAL	211	556	2184	17300	-	ınch	M-TOTAL	502	1464	4787	35218
Pore=0.125 inch N-Proc N-TC	154	248	571	1857				322	852	1755	7887
re-Irat	9.	302	1613	15443	ı	Ţ				3032	
Scale (Multiplied by base history)	1.75	1.50	1.25	1.00	;	Scale (Multiplied	by base history)	1.75	1.50	1.25	1.00
Neen Stress Blas	ហ្គ					Medic Offices Stas	1 1 3 1	0.0			

TABLE 16. SINGLE PORE VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS THICKNESS = 1.0 INCH ABS EH36

100 1nch	1nch
p N-FOTRL	N-TOTHL
95	209
11 275	578
11 1183	2351
19 9728	17621
Pore=3.300 inch N-Prop N-TOT 35 71 71 11 161 11	Pore::0.300 N-Prop 83 169 404
N-Init	M-Init
60	126
204	409
1022	1947
9289	16661
inch	inch
N-TOTAL	N-TOTAL
116	259
325	692
1356	2741
10928	20233
Pore=0.250 i	Forez0.250 i
N-Prop	t M-Prop
52	5 124
104	2 250
237	6 615
649	9 1844
M-Init	P.Init
64	135
221	442
1119	2126
10279	18389
Pores0,1875 inch	3 noch
k N-Prop N-TOTAL	N=TuTAL
85 158	35.4
27 170 422	32.8
5 387 1682	346.5
4 1066 13160	2479.2
orezil. 1875 N-Prop 85 170 387 1066	Furest 1875 in N-Prop (15) 203 427 429 1016 544
P Int P 73 252 1295	R-Intt 151 501 2449 21544
Scale (multiplied	Scale and highed
by base history)	by base history:
1.75	1.75
1.50	1.50
1.25	1.50
1.00	1.00
Meso Oftensis Bras.	Mean choose Brase (1831) (1931)

TABLE 17. UNIFORM POROSITY
VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS
THICKNESS = 0.5 INCH
ABS EH36

Pore=0.045 inch N-Init N-Prop N-TOTAL 16 1315 1331 47 3040 3087 Mon-propagating crack Non-propagating crack	Pore=0.045 inch N-Init N-Prop N-TUTAL Non-propagating crack Non-propagating crack Non-propagating crack
Pore=0.030 inch N-Init N-Prop N-TOTAL 28 2459 2487 Non-propagating crack Non-propagating crack	Pore=0.030 inch N-Init N-Prop N-TOTAL Non-propagating crack Non-propagating crack Non-propagating crack Non-propagating crack
Forestills anch	PorgeD.015 inch
N-Init N-Prop N-TUTAL	N-Init N-Prop N-TUTAL
Non-propagating crack	Non-propagating crack
Non-propagating crack	Non-propagating crack
Non-propagating crack	Non-propagating crack
Non-propagating crack	Non-gropagating crack
Scale (multiplied	Scale (Multiplied
by base history)	by base history)
1.75	1.75
1.50	1.50
1.25	1.25
1.00	1.00
Near Stress Blas	Mean Stress Bles
(ksi.)	(kst.)
6.5	0.0

TABLE 18. UNIFORM POROSITY
VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS
THICKNESS = 1.0 INCH
ABS EH36

4	N-TOTAL	282	1283	3369	14124	hnch	M-TOTHL	1894	crack	crack	crack
Poyest) (175 ; mrh	M-Prop N-TOTAL	575	1256	3265	13426	Pore=0.075 1nch	M-Prop	22 1872 1894	Non-propagating crack	Non-propagating crack	Non-propagating crack
۵	N-Init	10	27		698	ه د	M-Init	55	Mon-p	Non-pa	Mon-p
n D	N-TOTAL	1205	2765	9184	crack	nch	M-TOTAL	crack	crack	crack	crack
Poro=0 045 inch	N-Init N-Prop	1189	2718	6937	Mon-propagating crack	Pore=0.045 inch	N-Init N-Prop M-TOTAL	Non-propagating crack	Mon-propagating crack	Mon-propagating crack	Non-propagating crack
à	N-Init	16	4-	197	Mon-p	مّ	N-Inst	Mon-pi	Mon-pe	Non-pr	Mon-pr
Apar Sign Canada	N-Init N-Prop N-TOTAL	Non-propagating crack	Mon-propagating crack	Non-propagating crack	Non-propagating crack	Pore=0.015 inch	N-Init N-Prop N-TOTAL	Mon-propagating crack	Non-propagating crack	Non-propagating crack	Non-propagating crack
	by base history)	1.75	1.50	1.25	1.00	Scale (multiplied	by base history)	1.75	1.50	1.25	1.00
	(kg)	ند. د				Head Street Steel	cksin	0.0			

TABLE 19. CO-LINEAR POROSITY
VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS
NUMBER OF PORES = 3
THICKNESS = 1.0 INCH
ABS EH36

			ζ	ABS ENSO						
Hean Stress Blas	Scale (multiplied		Pores0.1875		g. + in T - M	Pore=0.250 i N-Prob	_	P-Init	Pore:0.300 i	_
(KS1)	1 25	27117 11	<u> </u>	119	99	58.	9.8	63	55	82
	. 50	253	ुक क	347	228	58		217	4	
	1.25	1900	215	1515	1161	131		1096	100	1196
	1.00	12147	530	12737	10710	359		10042	274	10316
Money Charles		۵	or#=0.1875	rech	<b></b>	ore=0.250	inch		Pore:0,300	inch
(163)		N-Init	N-Prop	N-TOTAL	N-Init	N-Prop		N-Init	N-Prop	N-TOTAL
0.0		152	111	263	139	69			51	184
	1.50	503	225	822	456	138			105	539
	200 H	2459	555	3016	2203	333	2536		241	2324
	1.00	21637	1760 2339	23397	19140	346			722	18697

TABLE 20. CO-LINEAR POROSITY
VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS
NUMBER OF PORES = 3
THICKNESS = 0.5 INCH
ABS EH36

1nch N-FOTHL 85 85 266 1248 10947	nnch R-TOTAL 184 548 2430 19771
Pore=0.250 1	Pore=0.250 1
14 N-Prop	1 N-Prop
36 19	19 45
38 38	16 92
31 07	13 2.27
10 237	10 631
R-Init	P-Int 139
66	139
228	456
1161	2203
10710	19140
inch N-TOTRL 111 330 1474 12623	290 290 290 292 2920 2920 23123
Pore≥0.1875 inch	Pore=0,1875 1
N-Prop N-T01	N.Prup
38	42
27	187
174 Iv	461
480 izv	1466
R-Int	N-Int 152
73	152
253	503
1500	2459
12147	:1637
111.th 15.5 145 1386 15410	100 h R -101 HL 254 968 3981 31268
Porv=0,125 inch	Porward, 125 1
R-frop R-fr	it Ne-Prop
68	30 174
138	37
313	37 948
956 B	33 948
PH-Inst	P-Tort
B7	180
307	607
1613	833
15444	27734
Scale (multiplied by lase Instory) 1.75 1.50 1.50 1.00	Scale implified by base in story? 1, 20 1, 50 1, 50 1, 50 1, 50 1, 00
Nean Stress Bras NSD 6.5	Mean Stress Blassikasi ikan nib

TABLE 21. CLUSTER POROSITY
VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS
THICKNESS = 0.5 INCH
ABS EH36

inch R-TUTAL 69 211 825 6129	inch N-FOTAL 214 534 1928 12949
Pore=0.300 inch N-Prop N-Tr 26 70 70 159 461	Pore=0.300 N-Prop 120 246 639 3625
N-Init 43 141 666 5668	N-Init 94 288 1289 10324
1nch N-TOTAL 217 526 1860 12609	1mch N-TOTAL 626 1531 5784
Pore=0.1875 inch N-Prop N-TG 151 303 745 12	PorceO.1875 inch N-Prop N-IC 487 1083 3658 gating crack
N-Init 66 223 1115 10119	Poreso N-Init N- 139 139 448 2126 Non-propagating
nch N-TOTAL 392 964 3486 23854	1168 3114
Porgeo, 125 inch N-Prop N-7 291 604 1585 5527	Pore=0.125 inch nit N-Prop N- 210 958 711 2403 Opegating chack Opegating chack
N-Init 101 360 1901 18327	P N-Init 210 711 Non-propaga Non-propaga
Scale (Multiplied by base history) 1.75 1.50 1.25 1.00	Scale (multiplied by base history) 1.75 1.50 1.25 1.00
Hean Stress Blas (ksi) 6.5	Mean Stress Blas

TABLE 22. CLUSTER POROSITY
VARIABLE AMPLITUDE FATIGUE LIFE PREDICTIONS
THICKNESS = 1.0 INCH
ABS EH36

inch	1nch
M-FOFAL	N-TOTAL
83	238
210	581
802	1862
5268	10908
Porc=0.400 inch N-Init M-Prop N-T01 37 46 119 91 551 251 688 55	Porg=0.400 inch N-Prop N-T0 157 336 788 i
M-Init	R-Init
37	81
119	245
551	1074
4580	8390
1rich	1mch
H-TOTAL	N-FOTRL
165	454
402	1061
1398	3437
9013	20603
Pore=0.250 1 N-Prop 113 229 259 557 1639	Pore=U.250 1 N-Prop 342 709 1820 7267
M-Init	H-Init
52	112
173	552
841	1617
7374	13336
1 nch	10ch
227	N- Cuthl
227	612
566	1452
1913	4705
12791	29401
Forest), 1875 inch	Pore 20, 1875 Inch
M-Frop M-TGI	R-Prop R- (0
160 5	9394 I
388 5	2523 A
770 13	10258 A
R-Toat 8.7 (2.58 ) 1148 10402	H-Init 140 140 453 2176 18643
Scale 'Multiphied	Scale (mu) trplaed
by Lusse history,	by base in story
1.76	1.75
1.50	1.50
1.25	1.26
1.00	1.00
Hean Stress Blas	Mean Otross Bras
(Ks) (C.5	(Ks) (0.0

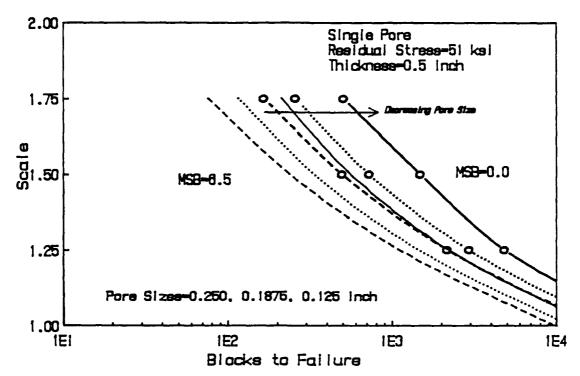


FIGURE 41. ENDURANCE CURVES FOR SINGLE PORES IN A 0.5-INCH THICK PLATE FOR SL-7 VARIABLE AMPLITUDE HISTORY CURVES CONNECTED BY CIRCLES REPRESENT A MEAN STRESS BIAS OF ZERO

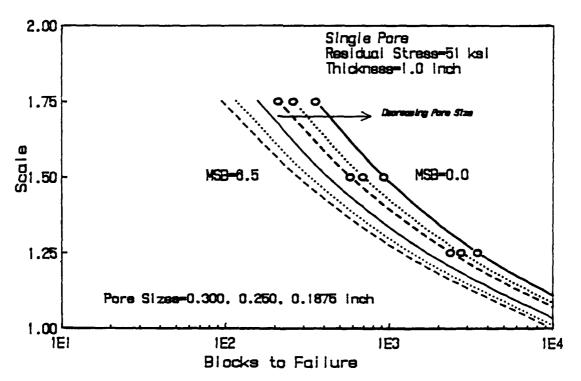


FIGURE 42. ENDURANCE CURVES FOR SINGLE PORES IN A 1.0 INCH THICK PLATE FOR SL-7 VARIABLE AMPLITUDE HISTORY. CURVES CONNECTED BY CIRCLES REPRESENT A MEAN STRESS BIAS OF ZERO

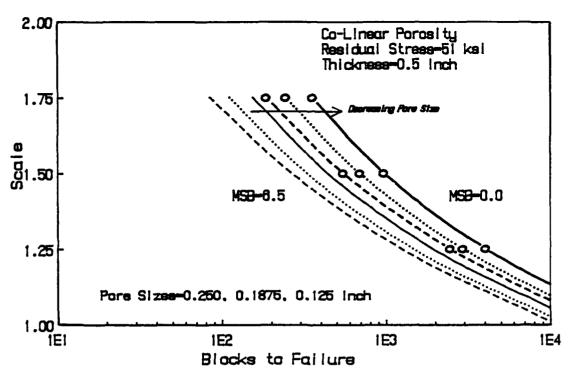


FIGURE 43. ENDURANCE CURVES FOR CO-LINEAR POROSITY IN A 0.5-INCH THICK PLATE FOR SL-7 VARIABLE AMPLITUDE HISTORY, CURVES CONNECTED BY CIRCLES REPRESENT A MEAN STRESS BIAS OF ZERO

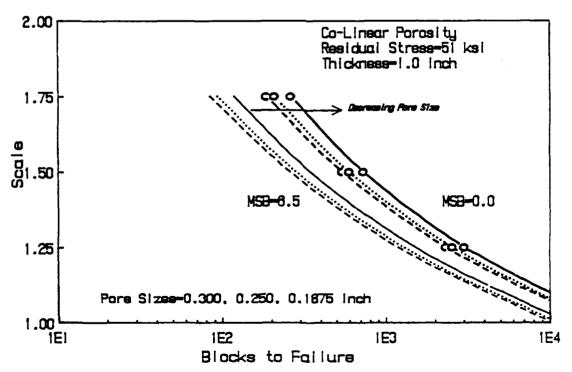


FIGURE 44. ENDURANCE CURVES FOR CO-LINEAR POROSITY IN A 1.0-INCH THICK PLATE FOR SL-7 VARIABLE AMPLITUDE HISTORY, CURVES CONNECTED BY CIRCLES REPRESENT A MEAN STRESS BIAS OF ZERO

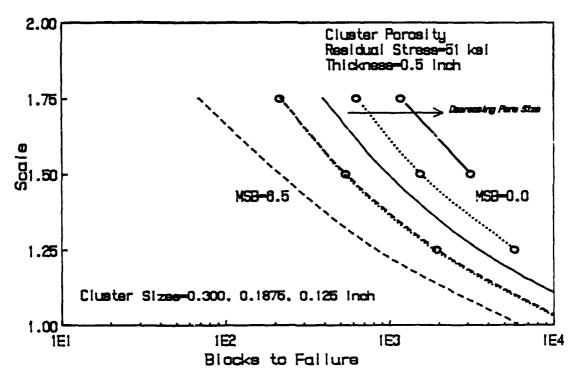


FIGURE 45. ENDURANCE CURVES FOR CLUSTER POROSITY IN A 0.5-INCH THICK PLATE FOR SL-7 VARIABLE AMPLITUDE HISTORY, CURVES CONNECTED BY CIRCLES REPRESENT A MEAN STRESS BIAS OF ZERO

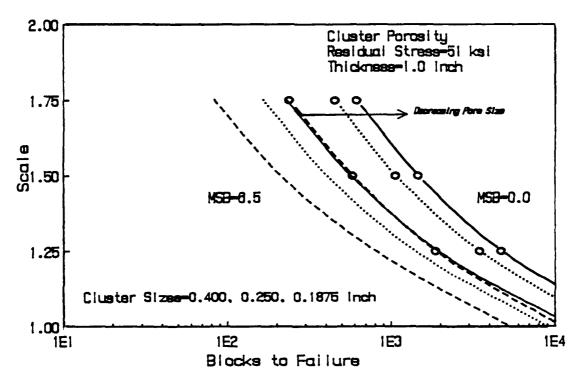


FIGURE 46. ENDURANCE CURVES FOR CLUSTER POROSITY IN A 1.0-INCH THICK PLATE FOR SL-7 VARIABLE AMPLITUDE HISTORY, CURVES CONNECTED BY CIRCLES REPRESENT A MEAN STRESS BIAS OF ZERO

# 9.1. Thickness

Two plate thicknesses were investigated in this study. It is important to note that since a specific width was not specified, the width of the plate is assumed to many times that of the plate thickness. The infinite width assumption means that the size of the porosity and subsequent crack are small in comparison to the plate and therefore the reduction in cross sectional area does not affect the nominal stress. The thickness of the plate, therefore, has no affect on the initiation life of the crack, all other parameters being equal. The difference in life between plate thicknesses is due to the propagation life. For equal pore sizes, it will simply take longer for a crack to grow toward the surface in a thicker plate. There is also a longer region where the stress intensity is not increased by the pore stress gradient or the back wall effect.

The fatigue life predictions proved to be relatively insensitive to the plate thickness. The larger thicknesses resulted in only slightly longer lives. This is due to the fact that life predictions are not greatly dependent upon the final crack length at failure (i.e., failure criterion and back surface effects). When the crack becomes large in size, the increased stress intensity drives the crack growth at an increasingly higher rate until failure occurs. Conversely, life predictions are very sensitive to initial crack lengths. See the initial crack length discussion in Section 7.1.

# 9.2. Residual Stress

As was noted in the literature survey, local residual stresses at the surface of pores is not reported. Masubuchi [22] indicated that tensile residual stresses as high as the yield strength of the base metal was measured near the centerline in butt welds. Two residual stress levels were used in the present study: the stress relieved condition (residual stress equals zero) and a residual stress equivalent to the yield stress in EH36 (51 ksi). The effect of residual stress is only accounted for in the initiation life calculations. Since the residual

stress field is thought to vary throughout the weld, accounting for the changing stress field in crack growth calculations would prove to be very complex. Therefore, the residual stress is taken as zero for all the propagation calculations.

For the initiation life calculations, a residual stress dictates the starting point for the loading. Figure 47 from Reference 10 illustrates the effect of the residual stress upon the stress-strain response of the material near the notch root of a weldment with reinforcement. An analogy can be drawn between the notch root material and the material near the surface of a pore since both act as geometrical stress concentrations or notches. The plot shows the stress-strain response for three materials; one strong, one tough, and one ductile; and the effect the residual stress,  $\sigma_{\bf r}$ , has on the set-up cycle. The result is a higher local mean stress than would be realized in the stress-free condition. The increase in mean stress is detrimental to fatigue life (see Section 9.3 Stress Ratio). Figure 48 shows the influence of residual stress on the fatigue life for a single pore as predicted by the model. Note the increase in life as residual stress is decreased.

# 9.3. Stress Ratio

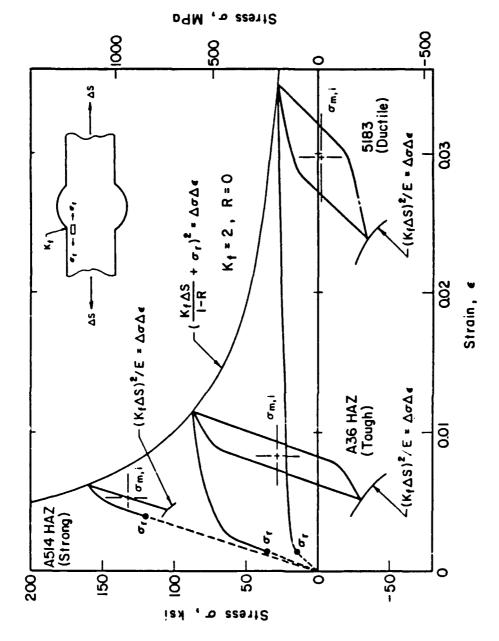
The stress ratio, defined as

$$R = S_{min} / S_{max}$$
,

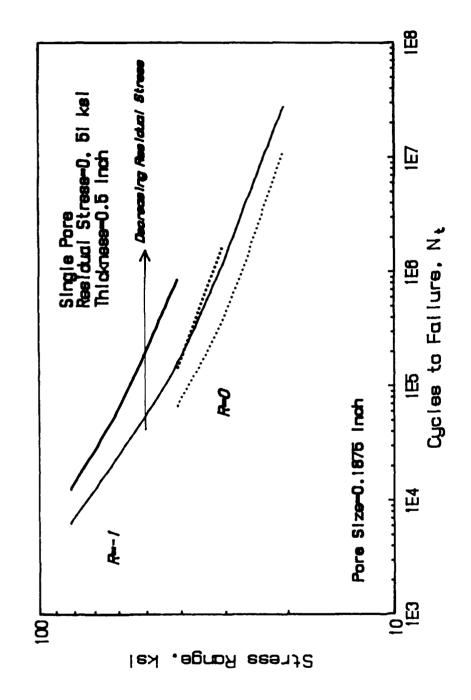
is incorporated into the model for both the initiation and propagation calculations. The stress ratio is directly related to the mean stress,  $S_{mean}$ , by

$$S_{mean} = \frac{S_{max}}{2} (1 + R)$$
 (20)

As the stress ratio increases, the tensile mean stress also increases. A tensile mean stress is generally observed to be detrimental for fatigue



SET UP CYCLE FOR ASTM 514 HAZ (STRONG), A36 HAZ (TOUGH) STEELS, AND ALUMINUM ALLOY 5183 WM (DUCTILE) MATERIALS. THE SET UP CYCLE RESULTS IN A TENSILE MEAN STRESS FOR THE STRONG AND TOUGH MATERIALS FIGURE 47.



S-N PLOT SHOWING THE TREND OF INCREASING FATIGUE RESISTANCE WITH DECREASING TENSILE RESIDUAL STRESS FIGURE 48.

life, provided that the strains are not great enough to cause complete mean stress relaxation. It can be seen from Equation 9,

$$\frac{\Delta \epsilon}{2} = \epsilon_f' (2N_f)^c + \left(\frac{\sigma_f' - \sigma_m}{E}\right) (2N_f)^b$$

that a tensile mean stress decreases the effective fatigue strength coefficient which is a measure of high cycle fatigue resistance. The strain-life equation is used to predict the initiation life at the pore surface, so a tensile mean stress will predict lesser initiation lives than zero or compressive mean stresses.

A high tensile mean stress is also found to increase crack growth rates. The crack growth rate relation,

$$\frac{da}{dN} = \frac{A \Delta K^{m}}{(1-R)}$$

was developed to account for the higher observed crack growth rates at higher stress ratios (and therefore higher mean stresses). Because both Equations 9 and 10 are used in the predictions, the trend on all of the S-N plots show a decreasing fatigue resistance with increased stress ratio.

The S-N plots show that none of the R = 0.5 predictions result in low lives ( $<10^5$ ). This seems to contradict the assertion that the high stress ratio loading is the most damaging. Actually this is the result of the method of choosing the stress levels for the predictions. Since the maximum stress for the predictions are chosen as 0.8, 0.6, 0.4, and 0.2 times the yield stress of the material, the stress ranges for the R = 0.5 are smaller than the other stress ratios. Stress range is the most influential parameter in the life prediction model. The small stress ranges in the R = 0.5 predictions therefore result in long lives.

#### 9.4. Pore Size

The influence of pore size affects both the crack initiation and propagation estimates. The fatigue notch factor,  $K_f$ , was developed to account for the observation that smaller notches were found to be less detrimental in fatigue than larger notches of similar geometry. The relation used in the model to account for this phenomenon (Equation 7),

$$K_f = 1 + \frac{K_t - 1}{1 + a/r}$$

was introduced by Peterson. It models the \*endency of larger pores to have lesser initiation lives.

The propagation lives are also affected by the pore size. The effective flaw size, once the crack initiates or sharpens, is defined as the sum of the pore radius and the emerging crack. The larger the pore size, therefore, the larger the initial crack size and shorter growth period required to reach the surface. The effect of decreasing pore size on fatigue life is noted on all of the S-N plots.

#### 9.5. Porosity Type

The effect of the type of porosity on fatigue life as predicted by the model can be inferred somewhat from Figure 49. The plot shows the stress ranges at total fatigue lives,  $N_{\rm t}$  of 10,000 for the four porosity types. This plot illustrates that the geometry or porosity type influences fatigue. In view of the assumptions made for each of the pore geometries, the uniform porosity geometry would be expected to have the greatest fatigue resistance, and the cluster geometry the least for equal pore sizes. For the larger pore sizes, the single pores would be expected to have only slightly more fatigue resistance than a co-linear arrangement of non-interacting pores of equal size. The infinite width assumption, where area percent porosity is not accounted for, is important to consider when making comparisons between the porosity types. For instance, the reduction in cross sectional area for the co-linear pores would result in

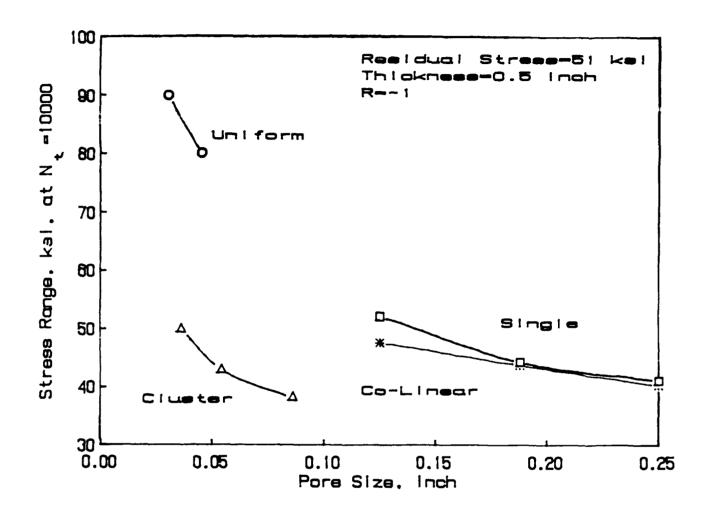


FIGURE 49. PLOT OF STRESS RANGE VS. PORE SIZE FOR THE FOUR TYPES OF POROSITY CONSIDERED IN THIS STUDY AT  $N_{\text{T}}$  = 10,000

a higher nominal stress, and the single and co-linear curves would be spread farther apart. If trends observed in this figure were extrapolated over the range of pore sizes, it is reasonable to assume that the single pore would show the greatest fatigue resistance, followed by the co-linear porosity, the uniform porosity, and the cluster porosity.

# 9.6. Relation to the Rules for Nondestructive Inspection of Hull Welds

The pore sizes chosen for the parametric study were based upon the Rules for Nondestructive Inspection of Hull Welds, 1986, prepared by the American Bureau of Shipping [54]. For uniform porosity, called "fine porosity" in the code, pore sizes less than 0.015 inch in diameter are not considered to be detrimental. This 0.015 inch pore was the smallest size examined in this study. For all the uniform porosity cases, the maximum allowed area percent porosity, 1.5 percent, was assumed. This pore size was generally found to have lives greater than  $10^8$  except at the highest stresses. The lowest predicted life for this pore size was 320,921 for fully reversed loading at a stress range of 81.6 ksi. Larger pore sizes were predicted to have decreasing fatigue resistance as seen in the S-N plots. These predictions indicate that the 0.015 inch pore size is a conservative value from a fatigue standpoint, for the minimum pore to be considered in design.

The largest isolated or single pore allowed in the code is 0.25 times the thickness of the plate, or 0.1875 inch, whichever is less. For the 0.5 inch-thick plate, the largest allowed pore is 0.125 inch. For the 1.0 inch-thick plate, the largest allowed pore is 0.1875 inch. Both of these maximum allowed pore sizes were predicted to have fatigue lives of about 10<sup>5</sup> for fully reversed loading at a stress range of 81.6 ksi, the worst case considered. Larger pores are predicted to have correspondingly lesser lives. The predictions indicate that these minimum values are again somewhat conservative and would not prove to be fatigue critical, at least for the material being considered.

The code also indicates that the concentration of porosity is not to exceed that shown in the charts in Figures 11 and 12. The fatigue

life predictions for clusters do indicate decreased fatigue life with increased pore concentration because of interaction. However, as discussed in Section 6.3, pores separated by a distance of two pore diameters do not affect the others stress field. The charts shown in Figures 11 and 12 would disallow pore separated by any less than five pore diameters. Again, this aspect of the code is conservative.

The assertion that the ABS code is conservative in its porosity allowables from a fatigue standpoint is not to be construed as an endorsement for its abandonment of even amendment. The presence of porosity, especially cluster porosity, in weld metal suggests improper welding practice and often masks other irregularities such as material degradation.

#### 10. SUMMARY

The aim of this study was to examine the effect of porosity upon the structural integrity of marine weldments. The parameters which influence the fatigue life of weldments with porosity were found from literature related specifically to porosity as well as traditional linear elastic fracture mechanics and low cycle fatigue concepts. Using this data, a model was developed to predict the fatigue lives of weldments with porosity and with reinforcement removed. Specific analysis routines were developed for life prediction of single pores, uniform porosity, colinear porosity, and cluster porosity. The model was used to predict the lives of a limited number of actual fatigue tests of welds containing severe clusters of porosity. The predictions agreed with the test results nearly within a factor of two. The model was used to examine the dependence of fatigue life on a number of parameters found to be influential. A variable amplitude loading history was developed using SL-7 stress history data. This history was used to generate variable amplitude life predictions for the four types of porosity being considered.

## 11. CONCLUSIONS

- (1) Porosity is not fatigue critical in butt weldments which have reinforcement intact. The stress concentration at the toe of the reinforcement is much more severe than internal porosity so fatigue cracks will initiate at the toe rather than a pore.
- (2) For butt welds with reinforcement removed, the following parameters have been found to influence fatigue life: material, thickness, residual stress, stress ratio, stress range, pore size and type of porosity.
- (3) In view of the assumptions made regarding pore geometry, for equal pore sizes, the single pore would be least detrimental in fatigue followed by co-linear porosity, and uniform porosity. Cluster porosity is predicted to be most detrimental.
- (4) For the SL-7 variable amplitude stress history, all pore geometries were predicted to last indefinitely. For members subjected to stresses 1.75 times that of the base history, lives on the order of tens of years were predicted.
- In relation to the findings of this study, the Nondestructive Inspection of Hull Welds, 1986, prepared by the American Bureau of Shipping, was found to be conservative from a fatigue standpoint. However, since the presence of porosity suggests improper welding procedure, other problems may with the weld may be present. The finding that the code is conservative from a fatigue standpoint is not sufficient reason for amendment of the porosity allowables.

#### 12. RECOMMENDATIONS FOR FUTURE WORK

To further substantiate the methodology presentated in this report, there is a need for more fatigue test data of weldment porosity. The authors were able to uncover only eight fatigue tests with sufficient documentation to which to apply the model. This sample is far from being statistically significant. It is recommended that a laboratory program be initiated investigate the models sensitivity to its various parameters.

A test program including a number of different ship steels and weld metals would prove insightful.

A method for predicting the three dimensional pore geomerty would greatly improve the usefulness of the proposed methodology. These life estimates were made with fracture surfaces showing the positional relationship of the pores. It would presently be difficult to determine the geometry from radiographs to predict fatigue lives of components prior to failure.

The problem of cavity interaction is not covered in any great depth in the literature. Interaction is a complex stress analysis problem perhaps best approached using photoelastic techniques. The availability of solutions to this problem would enhance the physical soundness of the methodology.

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# **APPENDIX**

STEP-BY-STEP EXAMPLE OF THE PREDICTIVE MODEL

#### **APPENDIX**

# Step-by-Step Example of the Predictive Model

# Single Pore

#### Parameters:

Stress range: 61.2 ksi

Stress ratio: -1

Residual stress: 51 ksi

Pore diameter: 0.1875 inch

Pore  $K_{+}$ : 2.054

Weld thickness: 1.0 inch

# Step 1. Notch analysis

The notch analysis determines the strains expected at the material adjacent to the pore surface. As discussed in Section 5.1.2, the fatigue notch factor is often used in place of the stress concentration factor when analysing fatigue loading. Solving for the material constant 'a' in Equation (8),

$$a = \left(\frac{300}{S_u}\right)^{1.8} \times 10^{-3} \text{ in.}$$
 (8)

using the ultimate strength of the ABS EH36 steel in Table 4 as 75 ksi, a = 0.01 inch. Using Equation (7),

$$K_{f} = 1 + \left(\frac{K_{t} - 1}{1 + a/r}\right) \tag{7}$$

and the values above, the fatigue notch factor,  $K_{\mathbf{f}}$ , is 1.95.

To determine the maximum and minimum strains at the pore surface due to cyclic loading, Nueber's rule is used. Because the loading is cyclic, the cyclic strength coefficient, K', and the cyclic strain hardening exponent, n', can be used in the final form of Equation (3),

$$\frac{\Delta S^2}{E} \qquad K_t^2 = \Delta \sigma \left( \frac{\Delta \sigma}{E} + \left( \frac{\Delta \sigma}{K} \right)^{1/n} \right)$$

The residual stress of 51 ksi is added to the left hand term giving,

$$\frac{(\Delta S K_t + \sigma_r)^2}{E} K_t = \Delta \sigma \left( \frac{\Delta \sigma}{E} + \left( \frac{\Delta \sigma}{K} \right)^{1/n} \right)$$

Solving for  $\Delta \sigma$ , the result is  $\Delta \sigma = 56.51$  ksi and  $\Delta \epsilon = 0.00716$ . reversal switches the coordinate axes of stress and strain, and the equation is solved again, this time without the added residual stress. This and all subsequent reversals use a value of the cyclic strength coefficient,  $K'_{rev}$ , equal to  $2^{(1-n')}*K'$ . This is necessary because K' is used to define the cyclic stress-strain curve which is constructed of the tensile hysteresis loop tips. The actual material stress-strain response during revesals follows a larger path when going into compression. The results for the reversal local stress range and strain range are 89.08 ksi and 0.00534. The minimum local stress is therefore -32.56 ksi and the minimum local strain is 0.0018. The local mean stress,  $\sigma_{\rm m}$ , is 11.97 ksi. Figure Al shows the hysteresis loop for the material at the pore surface for this loading case. Note that the residual stress state initially includes a large plastic strain value. In reality, the residual stress is generally below yield because at this stage the material stressstrain response follows the monotonic stress-strain curve. The fatigue life prediction model makes the assumption that the notch material assumes cyclic behavior relatively early in the loading history, so it is used throughout the analysis. The presence of the initial plastic strain does not affect the numerical computations in estimating the crack initiation life.

Step 2. Estimate cycles to initiation using low-cycle fatigue properties. Equation (9), the Coffin-Manson equation with Morrow's mean stress correction,

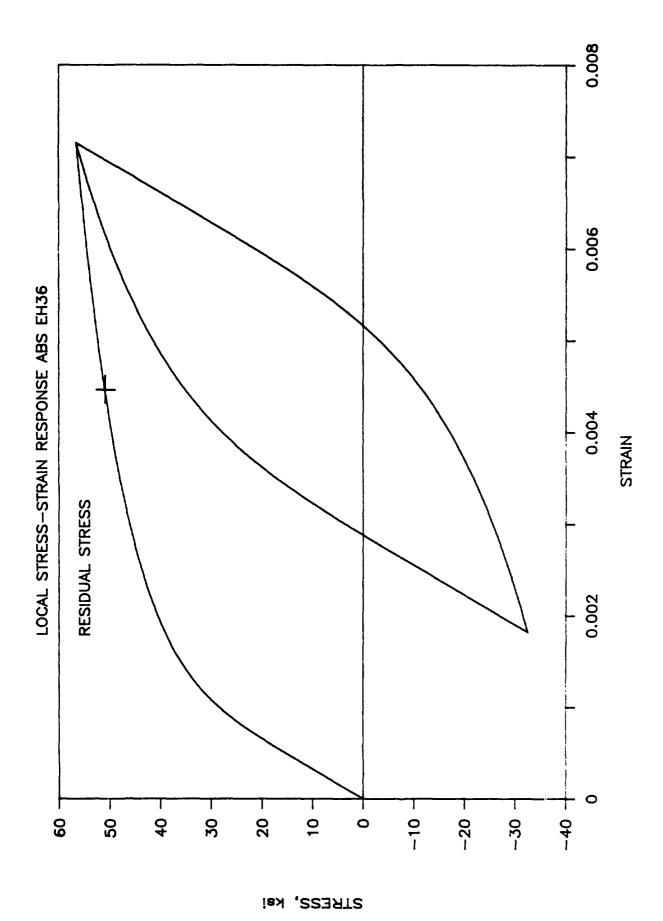


FIGURE A1. STRESS-STRAIN RESPONSE AT PORE SURFACE FOR EXAMPLE LIFE PREDICTION

$$\frac{\Delta \epsilon}{2} = \epsilon_{f}'(2N_{f})^{c} + \left(\frac{\sigma_{f} - \sigma_{m}}{E}\right) (2N_{f})^{b}$$
 (9)

is used to solve for the estimated cycles to failure,  $N_f$ . This again is an iterative procedure. For this example, the cycles to crack initation is 7971 cycles. The resulting  $N_f$  is actually the number of cycles required to initiate a fatigue crack at the pore surface since the calculated strains are local to this region. The remaining weldment is still intact at this cycle count. The rest of the analysis estimates the number of cycles to failure by crack propagation through the weldment.

Step 3. Estimate cycles required to propagate crack to failure.

The crack propagation model is outlined in section 5.2. The initial crack size assumption used throughout this study was 0.05 times the pore diameter. The initial crack size for this case is 0.0094 inch. To determine the stress intensity range for a given crack size and loading, the geometry correction factor from Equation (13)

$$Y = \frac{M_S M_t M_k}{\Phi_0}$$
 (13)

is calculated. When the crack is in the region of the stress concentration due to the pore, the stress intensity range solution is dominated by the stress gradient term,  $M_k$ . Calculating the  $M_k$  term requires a numerical procedure  $^{\left[47\right]}$  taking into account the stress gradient away from the pore. The  $M_k$  term is calculated by superposition of the notch stress gradient upon the crack. The expression is

$$M_k = \frac{2}{\pi} \frac{\sum_{i=1}^{n} \sigma_{bi}}{\sum_{i=1}^{n} (arcsin \frac{b_{i+1}}{a} - arcsin \frac{b_{i}}{a})}$$

where  $b_i$  is the position b along the crack,  $\sigma_{bi}$  is the stress at position  $b_i$  due to the notch (assuming no crack), and a is the crack length. In

this example, at the initial crack length of 0.0094 inch, the value of  $M_k$  is 2.11. The finite thickness correction factor,  $M_t$  is negligible (equal to one) at this small crack length. Also, the front surface term,  $M_s$ , is equal to unity for an internal crack. The crack shape factor,  $\Phi_0$ , for a circular crack is 1.57. The geometry correction factor, Y, is therefore 1.34 at the initial crack length. This value decreases rapidly with increasing crack length as shown in Figure 16. As the crack grows near to the surface, the value of Y begins to increase. For comparison, apply Equation 16 at a = t/2, the position of the crack front just before breaking the surface.  $M_t$  is 1.4, and  $M_k$  becomes near unity. The final value of Y is therefore 0.89.

Estimating the number of cycles to failure by crack propagation is accomplished by calculating the stress intensity factor range,  $\Delta K$ , at every cycle and incrementing the crack length according to the material crack growth rate. The estimated propagation cycles to failure for this example is 26722 cycles. The total estimated fatigue life is therefore 34693 cycles.

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